INTELLIGENT CONTROLLERS FOR VEHICLE SUSPENSION SYSTEM USING MAGNETORHEOLOGICAL DAMPER

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Dedicated:

To my lovely spouse

Siti Aishah Binti Mohd Daud

To my adorable son

Muhammad Khairun Nasirin Bin Mat Hussin

To my beloved parents

Allahyarham Ab Talib Bin Che Lin

Khadijah Binti Yusof

To my mother in law

Che Mas Binti Daud

And all family.

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ABSTRACT

Semi-active suspension control with magnetorheological (MR) damper is one of the most fascinating systems being studied in improving the vehicle ride comfort. This study aims to investigate the development of intelligent controllers for vehicle suspension system using MR damper, namely, the proportional-integral-derivative (PID) and fuzzy logic (FL) controllers optimized using particle swarm optimization (PSO), firefly algorithm (FA) and advanced firefly algorithm (AFA). Since the conventional optimization method always has a problem in identifying the optimum values and it is time consuming, the evolutionary algorithm is the best approach in replacing the conventional method as it is very efficient and consistent in exploring the values for every single space. The PSO and FA are among of the evolutionary algorithms which have been studied in this research. Nevertheless, the weakness of FA such as getting trapped into several local minima is an attractive area that has been focused more as a possible improvement during the evolutionary process. Thus, a new algorithm based on the improvement of the original FA was introduced to improve the solution quality of the FA. This algorithm is called advanced firefly algorithm. A parametric modelling technique known as Spencer model was proposed and employed to compute the dynamic behaviour of the MR damper system. The Spencer model was experimentally validated and conducted to capture the behaviour of the Lord RD-1005-3 MR damper with the same excitation input. A simulation of a semi-active suspension system was developed within MATLAB Simulink environment. The effectiveness of all control schemes were investigated in two major issues, namely the ability of the controller to reject the unwanted motion of the vehicle and to overcome the damping constraints. The result indicates that, the PID-AFA control scheme is more superior as compared to the PID-PSO, PID-FA, FL-PSO, FL-FA, FL-AFA and passive system with up to 27.1% and 19.1% reduction for sprung mass acceleration and sprung mass displacement, respectively. Finally, the performance of the proposed intelligent control schemes which are implemented experimentally on the developed quarter vehicle suspension test rig shows a good agreement with the results of the simulation study. The proposed control scheme of PID-AFA has reduced the sprung mass acceleration and sprung mass displacement over the FL-AFA and passive system up to 28.21% and 16.9%, respectively.

ABSTRAK

Kawalan ampaian separa-aktif dengan peredam magnetorheologi (MR) adalah salah satu bidang menarik yang dikaji dalam meningkatkan keselesaan semasa menaiki kenderaan. Penyelidikan ini mengkaji tentang pembangunan pengawal-pengawal pintar untuk sistem ampaian kenderaan dengan menggunakan peredam MR, iaitu pengawal terbitan-kamiran-berkadaran (PID) dan kawalan logik samar (FL) yang dioptimumkan dengan menggunakan pengoptimuman kawalan zarah (PSO), algorithma kelip-kelip (FA) dan algorithma kelip-kelip termaju (AFA). Oleh sebab kaedah pengoptimuman konvensional sentiasa mempunyai masalah dalam mengenal pasti nilai-nilai optimum dan ia juga memakan masa, algoritma evolusi adalah pendekatan yang terbaik untuk menggantikan kaedah konvensional kerana ia adalah sangat cekap dan konsisten dalam meneroka nilai untuk setiap ruang tunggal. PSO dan FA adalah antara algoritma evolusi yang telah dikaji dalam penyelidikan ini. Walaubagaimanapun, kelemahan yang ada pada FA seperti mudah terperangkap dalam beberapa minima tempatan adalah satu bidang menarik yang telah ditumpukan lebih sebagai peningkatan yang mungkin semasa proses evolusi. Oleh sebab itu, algoritma baru berdasarkan penambahbaikan daripada FA yang asal telah diperkenalkan bagi meningkatkan penyelesaian kualiti untuk FA. Algoritma ini dikenali sebagai algorithma kelip-kelip termaju. Satu teknik pemodelan yang berparameter dikenali sebagai model Spencer telah dicadangkan dan digunakan untuk mengira perilaku dinamik sistem peredam MR. Model Spencer telah disahkan dan dijalankan secara eksperimen untuk menangkap perilaku bagi peredam MR Lord RD-1005-3 dengan input pengujaan yang sama. Simulasi bagi sistem ampaian separa-aktif telah dibangunkan dalam persekitaran **MATLAB** Simulink. Keberkesanan bagi semua skim kawalan telah disiasat dalam dua isu utama, iaitu keupayaan skim kawalan untuk membuang pergerakan yang tidak diperlukan bagi kenderaan dan juga untuk mengatasi kekangan redaman. Keputusan menunjukkan bahawa kawalan PID-AFA lebih unggul berbanding dengan PID-PSO, PID-FA, FL-PSO, FL-FA, FL-AFA dan sistem pasif sehingga 27.1% dan 19.1% penurunan masing-masing bagi pecutan dan juga sesaran badan kenderaan. Akhirnya, prestasi bagi skim kawalan yang telah dilaksanakan secara eksperimen ke atas suatu rig ujikaji ampaian kenderaan sukuan telah menunjukkan keserasian dengan keputusan yang diperolehi melalui kajian simulasi. Skim kawalan yang dicadangkan bagi PID-AFA telah berkurang sehingga 28.21% dan 16.9% masing-masing bagi pecutan badan kenderaan dan sesaran badan kenderaan berbanding dengan FL-AFA dan sistem pasif.

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LIST OF ABBREVIATIONS

ABC Artificial bee colony

ACO Ant colony optimization

AFA Advance firefly algorithm

AFC Active force control

ANFIS Adaptive neuro fuzzy inference system

ΑI Analog input AO Analog output

D Derivative

DAQ Data acquisition

DE Differential Evolution

DOF Degree of freedom ER Electrorheological

EA Evolutionary algorithm

FA Firefly algorithm

FLC Fuzzy logic controller

FL fuzzy logic

FL-AFA fuzzy logic controller tuned using

advanced firefly algorithm

FL-FA fuzzy logic controller tuned using

firefly algorithm

GA Genetic algorithm

GC Gain coefficient

GRV Gain relative velocity

GSV Gain sprung velocity

HILS Hardware-in-loop-simulation

HSASF Hybrid Stability Augmentation System-Force

HTGA Hybrid taguchi genetic algorithm I - Integral

FL-PSO - Fuzzy logic controller tuned using

particle swarm optimization

LQR - Linear quadratic controller

MR - Magnetorheological

MSE - Mean square error

NF - Neuro-fuzzy

NI - National instrument

NPLDD - Non-parametric linearized data driven model

P - Proportional

PID - Proportional-integral-derivative

PID-AFA - Proportional-integral-derivative controller

tuned using advance firefly algorithm

PID-FA - Proportional-integral-derivative controller

tuned using firefly algorithm

PID-GA - Proportional-integral-derivative controller

tuned using genetic algorithm

PID-PSO - Proportional-integral-derivative controller

tuned using particle swarm optimization

PSO - Particle swarm optimisation

QFT - Quantitative feedback theory

RMSE - Root mean square error

SADFE - Semi active damping force estimator

SMC - Sliding mode control

RPSU - Road profile simulator unit

LIST OF SYMBOLS

Parameters of hysteresis loop model α Attractiveness β Attractiveness when r=0Fixed light absorption coefficient γ Filter time constant η Voltage input of first filter $\boldsymbol{\nu}$ Field dependent yield stress $\mu_i(x)$ Aggregated membership function The error function between the i^{th} particle with $\Delta Fitness$ the current fitness function ΔP Sum of two components ΔP_n Viscous pressure loss ΔP_{τ} Yield stress pressure loss \boldsymbol{A} Parameter controlling hysteresis amplitude A(t)State/system matrix Lower limit \boldsymbol{a} В Feedback gain B(t)Input matrix Upper limit b C_{D1} Dashpot C_{SKY} Skyhook damping coefficient C(t)Output matrix Viscous coefficient c

c_{1}, c_{2}	-	Acceleration constants
C_{o}	_	Viscous damping parameter
V		Proposed control system
CS	-	
C_s	-	Damping coefficient
D(t)	-	Feedthrough/feedforward matrix
e(t)	-	Error of input and output
F	-	Damping force
F_{SA}	-	Semi active skyhook damper force
F_d	-	Desired force estimated from control algorithm
F_n	-	Damping force at a given current supply
f_a	-	Actuator force
f_c	-	Frictional force
f_d	-	Semi active damping force
f_d G	-	Semi active damping force Forward gain
	- -	
G	- - -	Forward gain
G	- - - -	Forward gain Gap of fluid
G g I	- - - -	Forward gain Gap of fluid Desired current
G g I i, j	- - - - -	Forward gain Gap of fluid Desired current Number of particles
G g I i,j K_D		Forward gain Gap of fluid Desired current Number of particles Derivative gain
G g I i,j K_D		Forward gain Gap of fluid Desired current Number of particles Derivative gain Integral gain
G g I i, j K_D K_I		Forward gain Gap of fluid Desired current Number of particles Derivative gain Integral gain Proportional gain
G g I i,j K_D K_I K_P		Forward gain Gap of fluid Desired current Number of particles Derivative gain Integral gain Proportional gain Stiffness
G g I i,j K_D K_I K_P k k		Forward gain Gap of fluid Desired current Number of particles Derivative gain Integral gain Proportional gain Stiffness Accumulator stiffness
G g I i, j K_D K_I K_P k k k		Forward gain Gap of fluid Desired current Number of particles Derivative gain Integral gain Proportional gain Stiffness Accumulator stiffness Spring stiffness
G g I i, j K_D K_I K_P k k_D k_s k_t		Forward gain Gap of fluid Desired current Number of particles Derivative gain Integral gain Proportional gain Stiffness Accumulator stiffness Spring stiffness Tire stiffness

$m_{_{u}}$	-	Unsprung mass
m(t)	-	Control signal
p_{id}	-	$p_{\textit{best}}$ (particle personal best position)
p_{gd}	-	g_{best} (the global best position)
ps	-	Passive suspension system
Q	-	Flow rate
r	-	Distance between any two fireflies
r_1, r_2	-	Random number
и	-	Output first order filter
u(t)	-	Control/input vector
V_1	-	Sprung mass velocity (Skyhook)
V_{12}	-	Relative velocity (Skyhook)
V_{id}	-	Current velocity of the particle
V_s	-	Sprung mass velocity
V_t	-	Unsprung mass velocity
v_{st}	-	Relative velocity
W	-	Inertia weight/the pole width
W_i	-	Inertia weight
X	-	Damper displacement (Bouc-Wen model)
XD	-	Damper displacement (Spencer model)
x^*	-	Damper displacement
\dot{x}	-	Damper velocity
X_o	-	Initial damper displacement
X_i	-	First particle of firefly, i/initial displacement
X_{id}	-	Current position of the particle
x_f	-	Final displacement
x_j	-	Second particle of firefly, j

X_u	-	Unsprung mass displacement
\ddot{X}_u	-	Unsprung mass acceleration
X_s	-	Sprung mass displacement
\dot{x}_s	-	Sprung mass velocity
\ddot{x}_s	-	Sprung mass acceleration
x_r	-	Road profile displacement
x_1	-	Body displacement
\dot{x}_1	-	Body velocity
\ddot{x}_1	-	Body acceleration
x_2	-	Tire displacement
\dot{x}_2	-	Tire velocity
\ddot{x}_2	-	Tire acceleration
$x_1 - x_2$	-	Suspension deflection
x(t)	-	State vector
y_D	-	Damper displacement
y(t)	-	Output vector
z	-	Hysteretic variable

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CHAPTER 1

INTRODUCTION

1.1 Introduction

Global vehicle control systems are currently subjected to many important issues, not just in the present but also for the future. Controllability and stability of the vehicle is necessary to ensure the comfort of the driver and passengers. One of the essential issues is the ability of the suspension system to ensure that the vehicle body achieves a combination of good road handling, load carrying and ride comfort. The suspension system has its own mechanism which gives a great comfort and safety for driver and passengers inside the vehicle, especially when the vehicle hit a bump, a hole or taking a corner. The input disturbance, namely load disturbance and road disturbance always lead to a number of undesirable conditions for the vehicle system.

In order to have the best performance of the suspension, particular characteristics which are dealing with the regulation of body movement, the regulation of suspension movement and force distribution must be considered. There are several types of automotive suspension, namely, passive, semi-active and active suspension systems. Every each of these types has their own mechanisms and functions to act as the right contact between body and tires. However, the passive suspension system has a limitation to control the stability of vehicle which has motivated extensive researches on active and semi-active suspension systems (Aleyne and Hedrick, 1992).

Semi-active system contains a variable damping mechanism, such as an MR damper system, which is an alternative way to be either passive or active system. Due to the intelligent damper installation, a controllable system can be produced dynamically and the majority of the performance of a fully active system can potentially be achieved. For the active system, an additional actuator is installed into the system to produce an additional force for good vehicle performance. Although an active system produces a good ride quality and superior than the semi-active system, it has a major limitation such as the need for a large power source as well as being very expensive due to the high cost for design and development (Rao *et al.*, 2010; Krishnan, 2013).

In recent years, semi-active suspension system has become an attractive area of research. This is because the system can be as an active device without requiring a large power source. In addition, it can offer the reliability of passive devices (Tsang et al., 2006). Unlike passive system, variable damper of semi-active system can actually be effectively controlled in terms of damper stiffness based on required values in a particular situation. Generally, semi-active systems with variable damping force are divided into two types, namely, hydraulical and rheological. Among these, rheological damper is a current research attention due to the great advance in the rheological fluid development. Mainly, there are two types of rheological based damper which are magnetorheological (MR) and electrorheological (ER) dampers. Both principles are used to provide adjustable damping forces based on the alteration of the damping mediums' of viscosity (Fischer and Isermann, 2004).

The MR damper system has drawn significant attention from many researchers due to its fast time response, low power requirement, high dynamic range and mechanical simplicity (Koo *et al.*, 2004). There are four major areas of MR damper technology research including mathematical and numerical research (Yao *et al.*, 2009; Ye *et al.*, 2010), researches on fluid (Yang, 2002), researches on design and development (Fujitani *et al.*, 2003) and researches on MR damper control strategies (Caponetto *et al.*, 2003; Yao *et al.*, 2012; Khiavi *et al.*, 2013).

The semi-active MR damper model can be effectively used by considering the accuracy of the damper model and suitable controller for the damper. In the past few years, several promising control strategies have been introduced such as linear quadratic gaussian controller (Taylor *et al.*, 1999), H∞ controller (Du *et al.*, 2005) and linearly parameterized controller (Mori and Nilkhamhang, 2007). However, it could be realized that the alternative control algorithms can be introduced for further improvements.

1.2 Problem Statement

Vibration and road disturbance are always leads to a number of undesirable circumstances. The problem of automobile car when hitting the bump or any road disturbance for instance can lead to driver discomfort, restlessness and eventually fatigue. The semi-active and active suspension systems lead a major research area in automotive industries. However, due to the limitations of active suspension system such as high power consumption, expensive hardware and its complexity, semi-active suspension system is a possible way to be used in order to achieve the performance as good as a fully active system (Rashid *et al.*, 2007; Ubaidillah *et al.*, 2011)

One of the main issues needs to be addressed in the use of semi-active MR damper in the vehicle suspension system is how to design a suitable control strategy to overcome damping constraint and to reduce unwanted motion of body vehicle due to passing road profile disturbance based on existing MR damper model. This is due to the fact that, improper design of control scheme will lead the optimum target force is unpredicted. In addition, a proper design of control strategy is also important in overcoming the damper constraint by providing the same direction between target force and damper velocity. The advantages in the use of MR damper control strategy in improving the vehicle dynamics have been investigated extensively through simulation and experimental studies by many researchers such as Hudha *et al.*, (2008), Samin, (2010) and Kasemi *et al.*, (2012). In these previous works, the conventional method such as sensitivity analysis study was used to optimize the

parameter value of the control scheme. It is important to be mentioned that, the parameters of the control scheme tune using conventional method could not give the best results for the controller. This is due to the fact that, the process was not fully covered for every single space of data in achieving the minimum values of the objective function. Thus, the intelligent method based on evolutionary algorithm is the best approaches in replacing the conventional method for optimization process of the controller. The evolutionary algorithm is an efficient method which consistently able to explore every single space. While the conventional method resembles trial and error mechanisms, and depends on computational capacity, the evolutionary algorithm is a method that can learn faster as they run, and tend to be more intelligent and adaptive (Madić *et al.*, 2013).

The particle swarm optimization (PSO) and firefly algorithm (FA) are among of the evolutionary algorithms that could be used in this study. This is due to the fact that, these two algorithms have several advantages such as simple and easy to be used that leads many researchers to be focused more on these two algorithms (Bhushan and Pillai, 2013). In comparison, most of the previous research proves that, the performance of both PSO and FA are always competing each other in solving the linear and non-linear problems (Jones and Boizanté, 2011; Pal *et al.*, 2012). Nevertheless, the weakness of FA such as getting trapped into several local optima is an attractive area that could be focused more as a possible improvement during the evolutionary process. Thus, this is another issue that can be addressed in this study. A possible alternative in solving this problem is by introducing the scout position in firefly algorithm which can improve the quality solution of the original firefly algorithm. This algorithm is called advanced firefly algorithm (AFA).

The intent of the study is to attempt and explore the possibility of improving vehicle dynamics performance by introducing intelligent optimization techniques, namely, AFA, PSO and FA which is to be integrated with proportional-integral-derivative (PID) and fuzzy logic (FL) controllers by the use of semi-active MR dampers. The intelligent optimization techniques are effectively proposed to improve the accuracy of the parameter values for the proposed controllers instead the use of conventional optimization method.

The main works of this study include modelling of quarter vehicle suspension model, MR damper modelling, optimization algorithms, force tracking control and disturbance rejection control of semi-active MR damper system. This research is performed by first, through a simulation study of quarter vehicle and MR damper models using mathematical equations. Second, developments of proposed intelligent controllers are described to cancel out the road-induced disturbances effectively. Then, the research is continued with validation and evaluation of the proposed control scheme via experimental implementation in the platform of quarter vehicle suspension test rig and instrumented experimental vehicle.

1.3 Research Objectives

The main objectives of this study are described as the following:

- To develop an advanced firefly algorithm as a new intelligent tuning method
 for PID and FL based controllers in comparison with PSO and FA tuning
 methods for improvement of the body vehicle suspension system.
- 2. To model a semi-active quarter vehicle model integrated with the MR damper actuation system through simulation studies.
- 3. To evaluate the performance of the proposed intelligent control strategies via simulation evaluation and experimental validation.

1.4 Scope of the Study

The scopes of the research are as follows:

 In this study, simulation of two degree of freedom (DOF) quarter vehicle suspension system is developed via MATLAB Simulink environment. The parameters of the model are selected based real parameters on the quarter vehicle experimental rig developed in this research. The model can be assumed to have no slipping between tire and road surface and limited to vertical motion.

- 2. The MR damper characteristic is investigated in this study. A parametric approach using Spencer model is used to identify the actual damping force through the system based on force-velocity characteristic.
- 3. Intelligent control scheme are limited using PID and FL based controllers tuned using PSO, FA and AFA algorithms, and developed via simulation environment. The performances of the proposed control schemes are investigated in order to attenuate the unwanted effects of road disturbances and to improve ride comfort.
- 4. A quarter vehicle experimental rig completed with data acquisition (DAQ) and instrumentation system is developed. A semi-active suspension system with actuation system using the MR damper is integrated with the rig. The rig serves as a platform for the implementation of MR damper using the proposed control schemes.
- 5. The performance of the semi-active suspension system is evaluated based on the vertical body displacement, vertical body acceleration, suspension deflection and tire deflection only. Results are presented and analysed in time and frequency domains.

1.5 Research Contributions

A brief on the main contributions of this research is given as follows:

- 1. A new AFA is introduced to improve the weakness of the original FA and later to be compared with the FA and PSO.
- 2. This research gives details regarding the implementation of the AFA, FA and PSO tuned PID and FL controllers for the semi-active suspension system. This tuning method can be an alternative to the widely adopted heuristic or sensitivity analysis tuning method.
- 3. This research provides a detail development of the quarter vehicle suspension test rig with a small scale ratio and simple structure design. This experimental rig can be used as a platform to evaluate, validate and verify the performance of the proposed control schemes.

1.6 Significant of the Study

The main work of the suspension system is to support the vehicle weight and to isolate the vehicle body from any road profile disturbance. Thus, the semi-active MR damper is an alternative system that can be effectively used to improve the reliability of the passive system by implementing the active control device without considering very high power consumption. An appropriate controller is very important to overcome the damping constraint and to predict the optimum target force. Another important issue that can be addressed in this study would be the optimization method to identify the parameters of the controllers. It is said that, by using the conventional method approach, the parameters of the controllers could not give the best results since it was not fully covered for every single space values. Thus, it is important to study the intelligent optimization approach based on evolutionary method to improve the conventional method which is able to explore consistently for every single space values. It can also be noted that, the evolutionary method has the weaknesses that can be significantly addressed for the further improvement. Hence, it will be also a major investigation that can be focused in this study.

1.7 Research Methodology

This study begins with a literature review on the subject of interest related to the semi-active MR damper including MR damper modelling and MR damper control strategy. Since the optimization strategy based on evolutionary algorithm is one of the main interests in this study, an extensive review of the past research in different applications was also carried out.

The essential role in designing a semi-active suspension control depends on the accuracy of the semi-active damper model (Hudha, 2005). Thus, this study begins with the development of the MR damper model using a parametric Spencer model (Spencer *et al.*, 1997). The capability of the Spencer model with an appropriate parameter value is very important to predict the force-velocity and force-

displacement characteristics of the semi-active MR damper. To ensure that the Spencer model used in this study is able to capture the same dynamic behaviour with the real model of the semi-active MR damper, an experimental work was conducted. The behaviour of the semi-active MR damper was investigated based on the different voltage that can be sent to the damper coils. The performance of the closed loop force control was also studied theoretically.

This study proceeds with the development of the control strategy if the semiactive MR damper model has been established accurately. In this study, the
effectiveness of the PID and FL controllers to induce the disturbance rejection form
road profile were investigated. The PSO, FA and AFA algorithms were investigated
as an alternative optimization technique for tuning process of the proposed
controllers. Performance evaluation of the control strategies including the proposed
evolutionary algorithms were characterized by the ability of the proposed intelligent
controllers to cancel out the unwanted motion from road disturbance transmitted to
the body vehicle as well as their consistency in providing the same sign between the
target force and damper velocity. The controller strategies were evaluated on a
quarter vehicle model in both time and frequency domains.

The effectiveness of the control strategies with the best results in the simulation study to induce disturbance rejection were also followed by experimental validation using quarter vehicle test rig. The test rig was accomplished complete with its instrumentation system including the quarter vehicle mechanical system, electrical and electronics involving sensors (accelerometers, laser displacement sensor and force sensor) and PC-based interfacing, programming and control (software control using MATLAB Simulink). A road profile simulator unit was also part of the test rig that can provide the main disturbance element and to be controlled manually. Finally, the performance evaluation of the semi-active control system was performed on the quarter vehicle test rig. The proposed research strategy in the form of a flow chart is graphically shown in Figure 1.1.

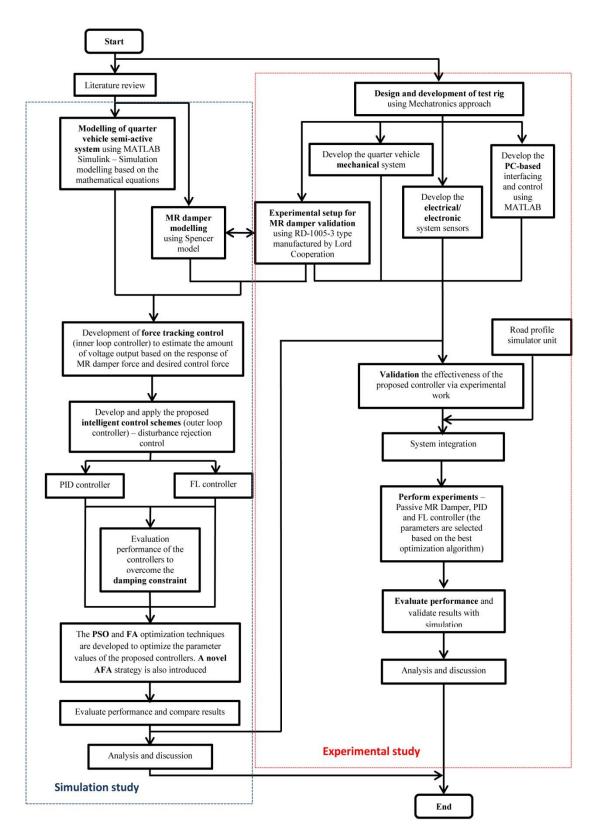


Figure 1.1 Research methodology flowchart

1.8 Thesis Outline

The thesis consists of seven chapters. Chapter 1 presents an introduction which includes an overview on the background of study, problem statement, research objectives and scopes. The contributions of the research, significance of the study, research methodology and the overall outline of this thesis are also presented.

Chapter 2 presents the literature review related to the subjects concerning this thesis. A review of suspension system, which includes vehicle classification as well as performance index of the MR damper system, is given thoroughly. In addition, brief overviews of the existing control strategies in all research areas are also highlighted in this chapter. Besides that, research gaps in the MR damper control strategy for vehicles' semi-active system is identified and discussed briefly.

Chapter 3 presents the simulation study of a quarter vehicle model and modelling of semi-active MR damper using a parametric Spencer model. An experimental work was also carried out to validate the Spencer model with the real damper model used in this study. Finally, model validation and force tracking control performance are also highlighted.

Chapter 4 describes an overview of the PID controller scheme concerning the road-induced disturbance rejection of semi-active system. The PID controller is first used to investigate the ability of this controller to overcome the damping constraint based on three different sinusoidal road profile disturbance (below sprung natural frequency, between sprung and unsprung natural frequencies and above unsprung natural frequency). Then, evolutionary algorithms based on the PSO, FA and AFA are described details as an alternative optimization technique to optimize the parameter values of the PID controller. A novel AFA as a new intelligent tuning method is also introduced in this chapter. Then, a new AFA is validated and tested using several benchmark functions in comparison with an original FA. The intelligent controller using PID tuned by PSO, FA and AFA are developed. The simulation study of three control schemes known as PID-PSO, PID-FA and PID-AFA are presented. The effectiveness of the semi-active suspension system and the

active control schemes of the vehicle model are defined. A comparative study is conducted to evaluate the performance of these three control schemes for improvement of ride comfort of the quarter semi-active vehicle system. All simulation results will be presented based on frequency and time domains analysis. The performance of the suspension system is evaluated in terms of body vehicle displacement, body vehicle acceleration and tire acceleration.

In chapter 5, another controller approach, namely, the FL control scheme is presented in this chapter. A general working principle of the fuzzy logic controller is highlighted. The effectiveness of the proposed controller to overcome damping constraint is also investigated. The optimization tuning algorithms using the PSO, FA and AFA are implemented in order to optimize the gain scaling of each input and output of the FL control scheme. Three control schemes known as FL-PSO, FL-FA and FL-AFA are presented. The performance of the proposed control schemes are compared to the passive suspension system. Finally, the advantages of the best evolutionary algorithm for both FL and PID control schemes over the passive system are also highlighted.

Chapter 6 presents the design and development of the experimental quarter vehicle suspension test rig equipped with the MR damper actuation system. The design of the vehicle system, measurement instruments and interfaces between the data signals and computer are further elaborated. The effectiveness of the control strategies with the best results in the simulation study are evaluated and validated experimentally. Comparative assessments of all controllers are presented. Finally, the limitation of the experimental study and differences over the simulation study is also highlighted.

Finally, Chapter 7 presents the conclusion of the research project. Recommendations of future works for further improvement are also discussed.

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