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# EFFECT OF CYLINDER DEACTIVATION STRATEGIES ON ENGINE PERFORMANCES USING ONE-DIMENSIONAL SIMULATION TECHNIQUE

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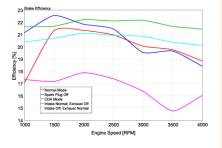
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## **Graphical abstract**



## Abstract

In order to meet consumer and legislation requirements, big investments on key technology strategies have been made to ensure fuel consumption is reduced. Recent technologies for gasoline engines are lean combustion technologies (including direct injection and homogenous charged compression ignition), optimizing intake and exhaust valve timing with valve lift and also cylinder deactivation system (CDA) have been practised to improve the engine efficiency. The purpose of this study is to investigate the engine behavior when running at different cylinder deactivation (CDA) strategies. One-dimensional engine model software called GT-Power is used to predict the engine performances. Five strategies were considered namely normal mode, spark plug off mode, cylinder deactivation mode, intake normal with exhaust off mode, and intake off with exhaust normal mode. Engine performance outputs of each strategy are predicted and compared at BMEP of 3 bars with engine speed of 2500 rpm. Also, the effect of CDA strategies on in-cylinder pressure and pumping loss are performed. The study shows that all of these cylinder deactivation strategies are capable of reducing the pumping loss (PMEP) and fuel consumption, thus increasing the thermal efficiency of the engine. The results suggest that the most beneficial strategy for activating CDA is for the case whereby both the intake and exhaust valves are kept closed. This CDA mode capable of increasing brake thermal efficiency up to 22% at entire engine speeds operation. This strategy successfully reduced the BSFC. It was found that most of these cylinder deactivation strategies improve the engine performance during part load engine condition.

Keywords: Cylinder deactivation, gt-power, intake, fuel consumption, pumping loss

# Abstrak

Dalam memenuhi keperluan pelanggan dan juga perundangan, pelaburan besar kepada strategi-strategi teknologi penting telah dilakukan untuk mempastikan penggunaan bahanapi dapat dikurangkan. Teknologi terkini untuk enjin petrol adalah teknologi pembakaran miskin (termasuk suntikan terus dan pencucuhan tekanan cas sekata), optima pemasaan injap masukan dan ekzos dengan angkatan dan juga sistem penyahaktifan silinder (CDA) telah digunapakai bagi meningkatkan kecekapan enjin. Dalam kajian ini, tujuannya adalah untuk mengkaji kelakuan enjin ketika beroperasi pada strategi penyahaktifan silinder yang berlainan. Perisian model enjin satu dimensi yang dikenali sebagai GT Power telah digunakan dalam meramal prestasi enjin. Terdapat lima jenis strategi yang dikaji iaitu mod biasa, mod padam palam pencucuh, mod penyahaktifan silinder, injap masukan aktif dengan injap ekzos nyahaktif, dan injap masukan nyahaktif dengan injap ekzos aktif. Keluaran prestasi enjin pada setiap strategi diramal dan dibandingkan pada beban BMEP 3 bar dengan putaran enjin 2500 rpm. Kesan strategi-strategi ini ke atas tekanan dalam silinder dan kehilangan daya pengepaman juga dilakukan. Kajian ini menunjukkan kesemua strategi-strategi penyahaktifan silinder ini telah memurunkan kehilangan pengapaman (PMEP) dan pengunaan bahanapi, malahan meningkatkan kecekapan haba enjin tersebut. Mod penyahaktifan silinder ini berupaya meningkatkan kecekapan haba enjin sehingga 22% pada keseluruhan halaju enjin. Hasil kajian ini mendapati strategi yang paling bermanfaat untuk penyahaktifan silinder adalah untuk kes di mana injap masukan dan ekzos ditutup keduanya. Strategi ini telah berjaya mengurangkan pengunaan bahanapi tentu (BSFC). Didapati, kebanyakan daripada strategi-strategi penyahaktifan silinder ini dapat meningkatkan prestasi enjin semasa dalam keadaan separa beban.

Kata kunci: Penyahaktifan silinder, gt-power, masukan, penggunaan bahanapi, kehilangan pengepaman.

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## **1.0 INTRODUCTION**

Deactivating cylinders is one of the proven method to reduce fuel consumption in a multi cylinder engine. The fuel improvement that benefit from deactivating cylinders ranging from 10% to 20% depending on the technological approaches [1-3].

Deactivation means that the cylinder is not producing combustion. There are several ways to deactivate the cylinder. The simplest method is just switching off the ignition. Since the fuel consumption is the main concern, the fuel injector must also be shut off. Other method includes shutting down the intake and exhaust valve from operating. In 1882, Mitsubishi Company tried several techniques of cylinder deactivation on its 1.4L, 4 cylinders Orion MD engine. The techniques are deactivate both intake and exhaust valves, or shut off the fuel supply while supplying fresh air without throttling, or shut off the fuel supply while re-circulating the exhaust gas, or just simply shut off the fuel supply. By doing so, it manage to reduce the fuel consumption up to 42% at certain engine condition [4].

This paper examines several options of deactivating the cylinders. The methods of deactivating the cylinders will be implemented in a simulation engine model by using GT Power software. Different options of cylinder deactivation should affect the engine performance in different ways. Cylinder deactivation mainly focuses on reducing the pumping loss. Pumping loss is high at part load engine operation due to partially opened throttle valve [5]. This create negative pressure inside the intake manifold. The pumping loss should also be reduced by increasing the intake manifold pressure or by unthrottled operation [6].

Shutting down some cylinders operation force the working cylinders to do extra work as to produce the same amount of work as if all the cylinders are running. In order to produce more work, the working cylinders need more air. The throttle opening should be opened wider to allow air access to the cylinders. When the throttle opening is open wider, it will increase the pressure in the intake manifold. This will reduce the pumping loss by the engine as the pressure is pushed into the active combustion chambers [7].

If the intake and the exhaust valves are kept closed, there should be air trapped inside the cylinder. The enclosed air works like a pneumatic spring which is periodically compressed and decompressed without overall pumping work. Therefore, the parasitic losses of the dragged cylinders are reduced [8].

# 2.0 METHODOLOGY

#### 2.1 Model Validation

In this study, the strategies to deactivate the cylinders are investigated using 1-D simulation approach. A simulation engine model has been built based on 1.6L Spark Ignition engine (Figure 1). The simulation model is based on one dimensional analysis by using GT Power software. The engine model has been built starts from the intake airbox system until exhaust tailpipe system. This is to make sure that the constructed model represents the real engine condition. For the intake and exhaust systems, almost all components are modeled as pipes. In GT-Power, pipes are used to represent these systems as tubes and they are connected by junctions. The flow model involves the solution of Navier-Stokes equations, namely the conservation of continuity, momentum and energy equations. Detail about the flow model can be referred to GT-Suite manual [9].

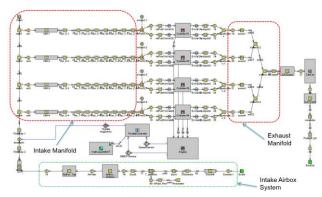


Figure 1 Engine model of four cylinders 1.6L engine.

The engine model has been constructed based on actual design, dimensions and parameters. This model has been validated with experimental data as shown in Figure 2 [10]. The errors between this simulation model and the actual engine testing are less than 5% which is acceptable to be used as a correlated model. This simulation engine model will be used to run the engine at several different modes.

### 2.2 Parametric Study

For this simulation study, there are four conditions of cylinder deactivation system to be analyzed. The conditions are based on the active and inactive of these four components which are the intake valve, exhaust valve, spark plug and fuel injector. The simulation is run in several modes which are in Normal mode and deactivated cylinders modes. There are four conditions of deactivating the cylinders:

- a) Turning off the spark plug
- b) Cylinder deactivation mode (CDA Mode)
- c) Intake valves close; exhaust valves normal
- d) Intake valves normal; exhaust valves close

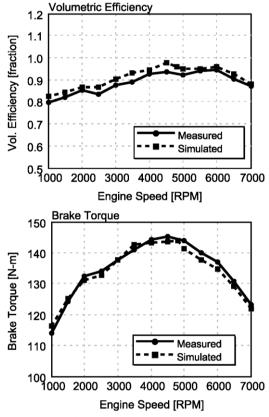


Figure 2 Comparison between simulated and measured data [10].

Normal mode refers to the normal operating conditions without any modifications to the original 4cylinder engine model. "Spark plug off" condition is when only the spark plugs from the deactivated cylinders are switched off as the engine valves are operating in normal condition. "Cylinder deactivation" (CDA) mode is when both intake and exhaust valves are switched off. As for the intake valves close; exhaust valves normal, the intake valves are switched off by setting the lift arrays to zero while the exhaust valves runs normally and vice versa. Table 1 summarizes the engine operating modes to be simulated. Note that all of these modes only affects cylinder 2 and 3. Cylinder 1 and 4 are allowed to operate normally without any modification. The performance output of the engine in normal and CDA mode are evaluated based on engine speed range between 1000 to 4000 rpm and at specific engine load which is 3 bar BMEP. This operating conditions are selected based on the common driving conditions in Malaysia [11, 12].

Table 1 Summary of engine modes to be simulated.

| Modes   | BMEP<br>(bar) | Intake<br>Valve | Exhaust<br>Valve | Spark<br>Plug | Fuel<br>Injection |
|---|---------------|-----------------|------------------|---------------|-------------------|
| Normal mode   | 3             | On              | On               | On            | On                |
| Spark plug off                                      | 3             | On              | On               | Off           | Off               |
| CDA Mode  | 3             | Off             | Off              | Off           | Off               |
| Intake valves<br>close; exhaust<br>valves normal    | 3             | Off             | On               | Off           | Off               |
| Intake valves<br>normal;<br>exhaust valves<br>close | 3             | On              | Off              | Off           | Off               |

### 3.0 RESULTS AND DISCUSSION

#### 3.1 At Full Load Condition

Comparison of engine brake power for each mode at full load condition is depicted in Figure 3. It shows that by deactivating two cylinders, the power significantly drops to half compared to the power that the normal mode can achieve. The best power curve among the deactivated modes comes from CDA mode where it produces a slightly higher power compared to other deactivated modes. The highest brake power CDA mode can produce is 32.3 kW running at 6500 rpm. The least production of power is when *intake normal; exhaust off* mode. This mode provides a steady curve between 4 kW to 20 kW of power between 1000 rpm to 4000 rpm.

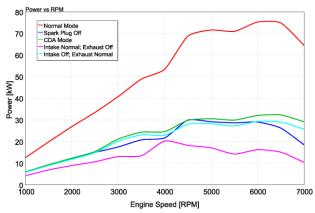


Figure 3 Brake power versus engine Speed for all engine modes at full load condition.

Figure 4 shows the brake torque produced at different engine speeds for all engine modes. Based on the graph, it is obvious that normal mode produced the highest torque among all the modes. The maximum torque generated by normal mode in this simulation is 145.8 Nm at 4500 rpm. The deactivated mode that produces the highest torque is CDA mode. CDA mode manages to produce a 67.3 kW of torque running at 3000 rpm. The deactivated modes produce almost half amount of torque of the normal mode. However, these modes are usable and recommended when low torque driving conditions are necessary. Such conditions are during highway cruising just to maintain vehicle speed whereby hard acceleration is not needed.

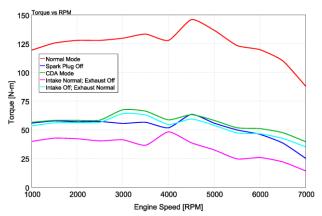


Figure 4 Brake torque versus engine speed for all engine modes at full load condition.

#### 3.2 At Part Load Condition

This study is to investigate the strategy of deactivating the cylinder especially at part load condition. Thus, engine simulation model is applied to predict the engine performance at several fixed variables. Such variables are:

- a) The target engine BMEP for each mode is 3 bar at every engine speed (part load condition).
- b) The deactivated parameters include the spark ignition, fuel injectors, and intake and exhaust valves.

Figure 5 shows the Net IMEP at cylinder #1 for different deactivation mode. The best engine condition to produce high IMEP is when *intake normal*; exhaust off mode is initiated. At 2500 rpm, *intake normal*; exhaust off mode manages to produce an IMEP value of 10bar which is 150% increase compared to *normal* mode that only produced 4 bar per active cylinder. *Intake off; exhaust normal* mode manages to produce 8.7 bar IMEP followed by *spark plug off* mode and CDA mode with 8.5 bar and 8.1 bar IMEP respectively at 2500 rpm. Each mode contributes more than 100% increase in IMEP when two cylinders are deactivated.

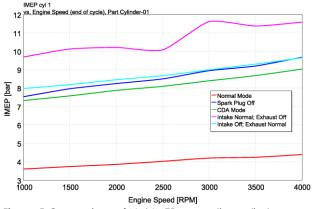


Figure 5 Comparison of Net IMEP per active cylinder versus Engine Speed for different deactivation strategy.

Increase in net IMEP can be related to the reduction in PMEP. The following formula shows the relationship between PMEP and IMEP:

Net IMEP = gross IMEP - PMEP

By applying cylinder deactivation modes, the amount of PMEP is highly reduced since the intake pressure is increased (

Figure **6**). Therefore, this produces more positive work to the engine with low pumping work.

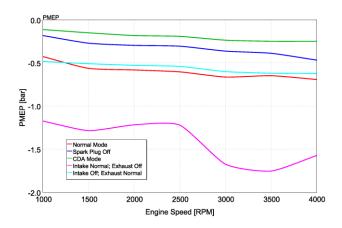


Figure 6 PMEP versus engine speed.

The main purpose of cylinder deactivation is to reduce pumping work in the engine. Therefore, it is important to evaluate the PMEP of the engine to determine its efficiency in producing positive work.

The engine model that produces the lowest pumping loss is the CDA mode where the intake and exhaust valves are both closed (

Figure **6**). By closing the intake and exhaust valves, the trapped air act like pneumatic spring as the piston move up to compressed it. This will reduce the pumping work done by the engine.

However, the intake normal; exhaust off mode shows higher pressure value of PMEP. This indicates

that this mode has high pumping loss. It happens due to the working intake valves in this mode while the exhaust valve is closed. Air is sucked into the cylinder during intake stroke, adding fresh air to the existing trapped air inside the cylinder that could not escape due to closed exhaust valve. This caused the pressure in the cylinder to build up and need extra work to compress the air.

LogP-LogV diagram is plotted for different modes of engine and it is shown in Figure 7. It is clear that all deactivated modes reduce the pumping loss by increasing the pressure in the active cylinders. All the deactivated modes show significant increase in pressure during compression and power stroke. In terms of positive work, the spark plug off mode produced a larger surface area in the graph during compression and power stroke. Therefore, it produced more work compared to the other engine modes. Overall, most of the deactivated modes shows significant reduction of pumping loss and increase of cylinder pressure for combustion.

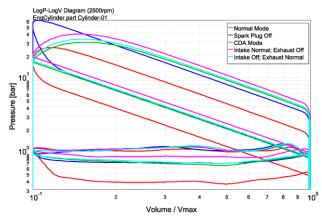
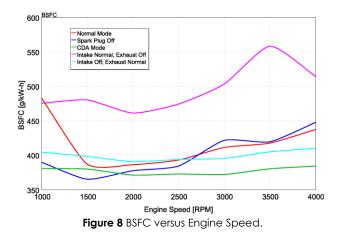


Figure 7 LogP-LogV diagram of all the engine modes (2500 rpm; BMEP: 3 bar)

BFSC is an important parameter in order to identify the fuel efficiency and fuel consumption of the engine. Based on the graph shown in Figure 8, the lowest BSFC recorded is 365 g/kW-h at engine speed of 1500 rpm. This happen when the engine operates with only the spark plug at cylinder #2 and #3 are switched off. However, the BSFC increased when higher engine speed are applied.

The CDA Mode shows a small fluctuating value of BFSC between 371 g/kW-h and 384 g/kW-h. The worst performance in BSFC is when the spark plug, fuel injection, and exhaust valves of cylinder #2 and #3 are switched off while the intake valves operates in normal condition which produced 558 g/kW-h at 3500 rpm. This can be related with the very high pumping pressure in the cylinder due to the opening of the intake valve which leads to very high fuel consumption to power the engine.



#### The graph in

Figure **9**, above shows the brake efficiency of the engine in different engine modes. *Normal* mode starts to increase in brake efficiency from 17% to 21.3% between 1000 rpm and 1500 rpm. CDA Mode produces steady brake efficiency between 1000 rpm to 4000 rpm with an average value of 21.8%. CDA mode is clearly the best in brake efficiency among all the modes.

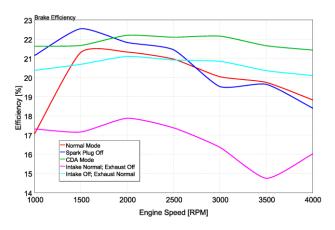


Figure 9 Comparison of engine's brake efficiency for every engine modes

#### 3.3 Summary of the Results

Table 2 and Table 3 shows the results summary between *normal* mode and the *cylinder deactivation* modes. These results are based on part load condition when the engine operates at 2500rpm, BMEP of 3 bar, and AFR of 13.8. 
 Table 2 Comparison between normal mode and the other cylinder deactivated modes.

| Modes  | Normal<br>Mode   | Spark<br>Plug Off | CDA<br>Mode      | Intake<br>Normal;<br>Exhaust<br>Off | Intake<br>Off;<br>Exhaust<br>Normal |
|--|------------------|-------------------|------------------|-------------------------------------|-------------------------------------|
| Net IMEP @ cyl 1                                     | 4.0              | 8.5               | 8.1              | 10.0                                | 8.7                                 |
| Net PMEP @ engine                                    | -0.6 bar         | -0.3 bar          | -0.2 bar         | -1.2 bar                            | -0.5 bar                            |
| BSFC @ engine  | 393<br>g/kW-h    | 384<br>g/kW-h     | 373<br>g/kW-h    | 474<br>g/kW-h                       | 394<br>g/kW-h                       |
| Brake Efficiency @ engine                            | 21%              | 21.4%             | 22.1%            | 17.4%                               | 21%                                 |
| Volumetric<br>efficiency @ cyl 1                     | 0.35             | 0.67              | 0.65             | 0.78                                | 0.69                                |
| Maximum Cyl.<br>Pressure @ cyl. 1                    | 25.4 bar         | 59.5 bar          | 30 bar           | 38.7 bar                            | 33 bar                              |
| Total Fuel<br>Consumption per<br>active cyl @ cyl. 1 | 13.1mg/c<br>ycle | 25.5<br>mg/cycle  | 24.7<br>mg/cycle | 29.6<br>mg/cycle                    | 26.1<br>mg/cycle                    |

**Table 3** Percentage differences between normal mode andthe cylinder deactivation modes.

| Modes  | Normal<br>Mode   | Spark<br>Plug<br>Off(%) | CDA<br>Mode(%) | Intake<br>Normal;<br>Exhaust<br>Off(%) | Intake Off;<br>Exhaust<br>Normal(%) |
|--|------------------|-------------------------|----------------|--|-------------------------------------|
| Net IMEP @ cyl. 1                                    | 4.03 bar         | 111.4                   | 101.5          | 148.8                                  | 116.4                               |
| Net PMEP @<br>engine                                 | -0.6 bar         | -50.0                   | -66.7          | 100.0                                  | -16.7                               |
| BSFC @ engine  | 393<br>g/kW-h    | -2.3                    | -5.1           | 20.6                                   | 0.3                                 |
| Brake Efficiency @<br>engine                         | 21%              | 1.9                     | 5.2            | -17.1                                  | 0.0                                 |
| Volumetric<br>efficiency @ cyl. 1                    | 0.35             | 91.4                    | 85.7           | 122.9                                  | 97.1                                |
| Maximum Cyl.<br>Pressure @ cyl. 1                    | 25.4 bar         | 134.3                   | 18.1           | 52.4                                   | 29.9                                |
| Total Fuel<br>Consumption per<br>active cyl @ cyl. 1 | 13.1mg/c<br>ycle | 94.7                    | 88.5           | 126.0                                  | 99.2                                |

## 4.0 CONCLUSION

Computer simulation techniques are applied to obtain better understanding in term of cylinder deactivation technology on engine performance. 1D engine model of 4-cylinders 1.6-liter SI engine has been constructed. The model has been well correlated with measured data. The correlated engine model has successfully predicted engine performance at various deactivation strategies. The simulation study shows that cylinder deactivation system in various modes does improve the engine in terms of efficiency and fuel consumption. As for reducing pumping loss or PMEP, the mode that is most effective and suitable is CDA mode where both the intake and exhaust valves are closed. CDA mode also has the lowest BSFC and overall fuel consumption amongst the other engine modes.

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