HIGH AMPLITUDE WAVE LINER EFFECTS ON JOURNAL BEARING PERFORMANCE

ASRAL

A thesis submitted in fulfilment of the requirements for the award of the degree of Doctor of Philosophy (Mechanical Engineering)

Faculty of Mechanical Engineering
Universiti Teknologi Malaysia

SEPTEMBER 2015

DEDICATION

To Sukmawati, Suci, and Farabbi for all their love and understanding

ACKNOWLEDGEMENT

Firstly, I would like to express my gratitude to my thesis supervisor, Assoc. Prof. Dr. Kahar bin Osman, for his valuable guidance, and encouragement.

Secondly, my thanks are due to the following individuals:

Rectors of Universitas Riau, Indonesia for guarantee my study leave at Universiti Teknologi Malaysia.

All Staffs and technicians of Fluids Laboratory for the assistance given during the experimental part of the work.

All my colleagues in the department with whom I had many fruitful discussions.

My dear parents, wife, and daughters who had to endure the difficulties resulting from my absence.

Finally, I wish to thank Local Government of Riau Indonesia for financial support this work.

ABSTRACT

Modification on the liner bearing was one of the ways to achieve sufficient lubricant on a journal bearing system. Efforts had shown that the ability to retain some lubricant by introducing wave like grooves on the liner has improved the performance of the bearings. However, the database for the modifications is still lacking. This study aims to establish and correlate modifications of the liner with the performance of the journal bearing. Numerical and experimental work were done to compare and relate several geometries of liner bearing modifications based on previous studies as well as new ones. The previously studied sine wave liner bearing involved investigations with the square and semi circular liners. Plain liner bearing was used as reference. All cases were investigated experimentally by a test rig under low operating loads of 30 N to 450 N with high speed conditions of 1200 RPM to 2800 RPM. Some of the parameters were validated in order to compare the numerical and experimental data. Case studies also included engine oil and palm olein as the lubricants. The performance of the bearings was analyzed by examining the side flow rates, lubricant temperature change, eccentricity ratio, and pressure distributions. The results show that, modifying the shape liner bearing under all operating conditions could increase the lubricant flow rate which was approximately 1.5 times than the plain liner bearing and reduce the lubricant temperature change by about 35%. The bearings with the wave shape liner led to the eccentricity ratio increase but within the acceptable range of 0.6 to 0.9. Majority of the results showed lower a maximum pressure than the plain liner bearing with the exception of the bearing with the square wave shape liner.

ABSTRAK

Pengubahsuaian ke atas pelapik galas adalah salah satu cara untuk mencapai pelincir yang mencukupi pada sistem galas jurnal. Usaha telah menunjukkan bahawa keupayaan untuk mengekalkan beberapa pelincir dengan memperkenalkan alur seperti gelombang pada pelapik telah meningkatkan prestasi galas. Walau bagaimanapun, pangkalan data untuk pengubahsuaian masih kurang. Kajian ini bertujuan untuk mewujudkan dan menghubungkaitkan pengubahsuaian pelapik dengan prestasi galas jurnal. Kaedah berangka dan eksperimen dilakukan untuk membandingkan dan mengaitkan beberapa geometri pengubahsuaian pelapik galas berdasarkan kajian terdahulu dan yang baru. Kajian sebelum ini meliputi galas dengan pelapik gelombang sinus, disiasat bersama-sama dengan pelapik bentuk gelombang persegi dan bulatan separa. Pelapik galas biasa telah digunakan sebagai rujukan. Semua kes-kes telah disiasat secara eksperimen di bawah beban operasi yang rendah sebanyak 30 N hingga 450 N dengan keadaan kelajuan tinggi sebanyak 1200 RPM hingga 2800 RPM. Beberapa parameter telah disahkan dalam usaha untuk membandingkan data berangka dan uji kaji. Kajian kes juga merangkumi minyak enjin dan minyak sawit sebagai minyak pelincir. Prestasi galas dianalisis dengan memeriksa kadar aliran sisi, kenaikan suhu minyak pelincir, kesipian dan agihan tekanan. Keputusan menunjukkan bahawa, mengubahsuai bentuk pelapik galas di bawah semua keadaan operasi, boleh meningkatkan kadar aliran minyak pelincir sebanyak lebih kurang 1.5 kali daripada pelapik galas biasa dan mengurangkan perubahan suhu minyak pelincir sehingga 35%. Galas dengan pelapik bentuk gelombang membawa kepada kenaikan nisbah kesipian tetapi dalam julat yang boleh diterima iaitu 0.6 hingga 0.9. Majoriti keputusan menunjukkan tekanan maksimum lebih rendah daripada galas pelapik biasa dengan pengecualian kepada galas dengan pelapik bentuk gelombang persegi.

TABLE OF CONTENTS

CHAPTER		TITLE	PAGE
		CLARATION	ii
		DICATION	iii
		KNOWLEDGEMENTS	iv
	ABS'	TRACT	V
	ABS	TRAK	vi
	TAB	LE OF CONTENTS	vii
	LIST	T OF TABLES	xi
	LIST	OF FIGURES	xiii
	LIST	T OF SYMBOLS	xxxi
	LIST	T OF APPENDICES	xxxiii
1	INTI	RODUCTION	1
	1.1	Background of the Problem	1
	1.2	Problem Statement	4
	1.3	Objectives	4
	1.4	Scope of Research	5
	1.5	Research Contributions	6
	1.6	Outline of Thesis	6
2	LITI	ERATURE REVIEW	8
	2.1	Brief Overview	8
	2.2	Basic Concepts in Journal Bearing Studies	10

				viii
		2.2.1	Liner Bearing	11
		2.2.2	Loads	11
		2.2.3	Lubricants	11
		2.2.4	Journal Bearing Eccentricity	12
		2.2.5	Ratio Length to Diameter	13
		2.2.6	Lubrication Regimes	14
		2.2.7	Side Flow Journal Bearing	15
		2.2.8	Minimum Film Thickness	17
		2.2.9	Maximum Film Pressure	19
	2.3	Lubrio	cants Selection and Lubrications Studies	20
	2.4	Thern	nal Characteristic of Journal Bearings	25
	2.5	Pressu	are Distribution within Journal Bearings	30
	2.6	Non-C	Circular Journal Bearings	32
	2.7	Groov	ved Journal Bearings	35
	2.8	Nume	erical Studies of Journal Bearings	37
	2.9	New (Gap for Current Study	42
3	MET	THODO	LOGY	46
	3.1	Gener	ral	46
	3.2	Lubrio	cants	47
	3.3	Exper	imental Setup	49
		3.3.1	Wavy Liner Bearing Concepts	52
		3.3.2	Fluid Film Pressure Measurement	63
		3.3.3	Fluid Film Thickness Measurement	66
		3.3.4	Oil Flow Side Measurement	68
	3.4	Nume	rical Approach	71
		3.4.1	Mesh Assessment	71
		3.4.2	Flow Modeling	72
		3.4.3	Boundary Conditions	74
		3.4.4	Parameters of Iteration	75
		3.4.5	Determination of Flow Type	76
		3.4.6	Selection of Model	77
		3.4.7	Newtonians Order of Lubricants	79
		3.4.8	Validation	80

4	RES	ULTS A	ND DISCUSSION	82
	4.1	Gener	al	82
	4.2	Oil Si	de Flow Rate of Journal Bearing	83
		4.2.1	Oil Side Flow Rate under Constant Speed	84
		4.2.2	Oil Side Flow Rate under Various Speeds	86
	4.3	Lubric	cants Temperature Changes	91
		4.3.1	Lubricants Temperature Changes under	
			Constant Speed	91
		4.3.2	Lubricants Temperature Changes under	
			Various Speeds	94
	4.4	Journa	al Bearing Eccentricity	98
		4.4.1	Eccentricity under Constant Speed	99
		4.4.2	Eccentricity under Various Speeds	102
	4.5	Pressu	are Distribution	106
		4.5.1	Pressure Distribution Circumferentially of	
			Journal Bearing	106
		4.5.2	Peak of Maximum Pressure under	
			Various Speeds	114
	4.6	Data V	Validation	119
		4.6.1	Clarification of Negative Pressure	121
		4.6.2	Contours of Pressure on Journal bearing	123
		4.6.3	Comparisons of Pressure Profile	130
	4.7	Detail	s in Various Number and Amplitude of wave	
		Liners	Study on Journal Bearing Performance	150
		4.7.1	Square Wave Liner with Various Numbers	
			of Wave Journal Bearing Performance	150
		4.7.2	Semi Circular Wave Liner with Various	
			Amplitudes of Wave Journal Bearing	
			Performance	156
	4.8	Summ	nary of Result and Discussion	162
		4.8.1	Summary of Oil Side Flow	162
		4.8.2	Summary of Lubricant Temperature	
			Changes	163
		4.8.3	Summary of Journal Bearing Eccentricity	165

$^{\Lambda}$	

		4.8.4 Summary of Pressure Distribution	165
5	CON	ICLUSIONS AND RECOMMENDATIONS	167
	5.1	Conclusions	167
	5.2	Recommendation	169
REFERE	NCES		170
Appendice	es A – B		181 - 203

LIST OF TABLES

TABLE NO.	TITLE	PAGE
2.1	Summary of previous studies focused on modification of the liner bearing in order to find the gap for current study	42
3.1	Properties of Palm Olein and Engine Oil SAE 20W-40 as lubricants	48
3.2	Specification of main component the rig testing	52
3.3	Wave liner bearing specifications	59
3.4	Reynolds number calculation	77
4.1	Representation of data collection of wave journal bearing	82
4.2	Side flow rate of square wave liner bearing with various number of wave and loads under lubricants of palm olein (PO) and engine oil (EO)	151
4.3	Lubricant temperature change of square wave liner bearing with various numbers of wave and loads under lubricant of palm olein (PO) and engine oil (EO)	153

4.4	Eccentricity ratio of square wave liner bearing with	
	various numbers of wave and loads under lubricant	
	of palm olein (PO) and engine oil (EO)	154
4.5	Peak of maximum pressure of square wave liner bearing	
	with various numbers of wave and loads under lubricant	
	of palm olein (PO) and engine oil (EO)	155
4.6	Side flow rate of semi circular wave liner bearing with	
	various amplitudes of wave and loads under lubricant	
	of palm olein (PO) and engine oil (EO)	157
4.7	Lubricant temperature change of semi circular wave liner	
	bearing with various amplitudes of wave and loads under	
	lubricant of palm olein (PO) and engine oil (EO)	158
4.8	Eccentricity ratio of semi circular wave liner bearing with	
	various amplitudes of wave and loads under lubricant of	
	palm olein (PO) and engine oil (EO)	159
4.9	Peak of maximum pressure of semi circular wave liner	
	bearing with various amplitudes of wave and loads under	
	lubricant of palm olein (PO) and engine oil (EO)	161

LIST OF FIGURES

FIGURE NO	. TITLE	PAGE
2.1	Schematic of journal bearing component, Khonsari and Booser (2008)	10
2.2	Stribeck curve for types of journal bearing lubrication related to the bearing parameter, Mott (2004)	15
2.3	Raimondi and Boyd chart for flow variable, Shigley <i>et al.</i> (2004)	16
2.4	Raimondi and boyd chart for determining the ratio of side flow to total flow, Shigley <i>et al.</i> (2004)	17
2.5	Raimondi and Boyd chart for minimum film thickness variable and eccentricity ratio, Shigley <i>et al.</i> (2004)	18
2.6	Raimondi and boyd chart for determining the position of the minimum film thickness, Shigley <i>et al.</i> (2004)	18
2.7	Raimondi and boyd chart for determining the maximum film pressure, Shigley <i>et al.</i> (2004)	19
2.8	Raimondi and Boyd chart for finding the terminating	

	٠	
X	1	٦

		xiv
	position of the lubrication film and the position of	
	maximum film pressure, Shigley et al. (2004)	19
2.9	Comparison of palm oil dynamics viscosity to engine oil	
	SAE 20W under the effect of temperature increase	23
2.10	Characteristics of maximum temperature, power loss, and	
	flow rate in various specific loads of circular	
	journal bearing, Ma and Taylor (1996)	27
2.11	Characteristics of maximum temperature, power loss,	
	and flow rate in various specific loads of elliptical	
	journal bearing, Ma and Taylor (1996)	28
2.12	Lemon-bore bearing and model geometry,	
	Ostayen and Beek (2009)	29
2.13	Non-circular journal bearing types, (a) Offset-halves	
	journal bearing, and (b) Elliptical journal bearing,	
	Chauhan <i>et al.</i> (2011)	29
2.14	Three wave liner journal bearing, Dimofte (1995)	33
2.15	Oil flow rate profile of wave journal bearing in various	
	radial loads, Dimofte et al. (2007)	34
2.16	Journal bearings with one axial groove,	
	Singh et al. (2008)	37
3.1	Viscosity variations of palm olein and engine oil	
	as lubricants with temperature	48
3.2	(a) Real image of rig testing component, and	
	(b) The arrangement of rig testing components; oil tank 1,	

	holder 2, main frame 3, belt 4, electric motor 5, load 6,	
	pressure gage 7, pully 8, main shaft 9, dial gauge 10, journal bearing 11, support bearing 12, load lever 13	50
3.3	(a) Real image of bearing with pressure tap, and(b) Schematic of bearing testing	51
3.4	Basic geometry of journal bearing	54
3.5	Detail of triangle geometry for fluid film evaluation	54
3.6	Parameters description on square-wave liner bearing	56
3.7	Parameters description on sinusoidal-wave liner bearing	56
3.8	Parameters description on semi circular-wave liner bearing	57
3.9	(a) Sectional area of three different types of wave liner bearings, and (b) displays the real image of wave liner bearing	58
3.10	Geometry of semi circular wave liner bearing, (a) design, and (b) real image.	61
3.11	Geometry of square wave liner bearing, (a) designs, and (b) real image	62
3.12	Geometry of sinusoidal wave liner bearing, (a) designs, and (b) real image	63
3.13	Real image of pressure gage	65
3.14	Schematic of pressure measurement	66

		xvi
3.15	The manner of dial gauge placement on the bearing	67
3.16	Schematic of film thickness measurement	68
3.17	Schemes of lubricant side flow, (a) real image and (b) illustration	70
3.18	Sample of mesh assessment for circumferential pressure of journal bearing	72
3.19	Set of typical boundary condition	74
3.20	Displays of pressure contour of bearing calculated with, (a) laminar model, and (b) turbulence model	78
3.21	Comparisons of pressure distribution at the middle area of around circumference of bearing calculated with laminar and turbulence model	79
3.22	Pressure distributions around circumferential bearing as the variation of Power Law index, n	80
3.23	Sketch of circumferential and axial position	81
4.1	Side flow rate of plain liner in various loads and shaft speed of 1260 RPM constant	85
4.2	Side flow rate of various wave shapes liner with various loads and shaft speed of 1260 RPM constant	86
4.3	Side flow rate of plain liner in various shaft speeds under load condition of (a) 31.5 N, (b) 63 N, (c) 126 N, (d) 252 N, and (e) 441 N	89

• •	
X V/11	
71 V 11	

4.4	Side flow rate of various wave shapes liner in various shaft speeds under load condition of (a) 31.5 N, (b) 63 N, (c) 126 N, (d) 252 N, and (e) 441 N	90
4.5	Lubricant temperature change of plain liner in various loads and shaft speed of 1260 RPM constant	92
4.6	Lubricant temperature changes of various wave shapes liner in various loads and shaft speed of 1260 RPM constant	93
4.7	Lubricant temperature changes of plain liner in various shaft speeds under load condition of (a) 31.5 N, (b) 63 N, (c) 126 N, (d) 252 N, and (e) 441 N	97
4.8	Lubricant temperature changes of various wave shapes liner in various shaft speeds under load condition of (a) 31.5 N, (b) 63 N, (c) 126 N, (d) 252 N, and (e) 441 N	98
4.9	Eccentricity ratio of plain liner bearing in various loads and shaft speed of 1260 RPM constant	100
4.10	Eccentricity ratio of various shapes wave liner in various loads and shaft speed of 1260 RPM constant	101
4.11	Eccentricity ratio of plain liner bearing in various shaft speeds under load condition of (a) 31.5 N, (b) 63 N, (c) 126 N, (d) 252 N, and (e) 441 N	104
4.12	Eccentricity ratio of various shapes wave liner in various shaft speeds under load condition of (a) 31.5 N, (b) 63 N, (c) 126 N, (d) 252 N, and (e) 441 N	105
4.13	Pressure profiles at middle area around circumference	

		xviii
	of plain liner bearing under load condition of 31.5 N and shaft speed of 1260 RPM	108
4.14	Pressure profiles at middle area around circumference of plain liner bearing under load condition of 63 N and shaft speed of 1260 RPM	108
4.15	Pressure profiles at middle area around circumference of plain liner bearing under load condition of 126 N and shaft speed of 1260 RPM	109
4.16	Pressure profiles at middle area around circumference of plain liner bearing under load condition of 252 N and shaft speed of 1260 RPM	109
4.17	Pressure profiles at middle area around circumference of plain liner bearing under load condition of 441 N and shaft speed of 1260 RPM	110
4.18	Pressures profile at middle area around circumference of various wave shapes liner under load condition of 31.5 N and shaft speed of 1260 RPM	112
4.19	Pressures profile at middle area around circumference of various wave shapes liner under load condition of 63 N and shaft speed of 1260 RPM	112
4.20	Pressures profile at middle area around circumference of various wave shapes liner under load condition of 126 N and shaft speed of 1260 RPM	113
4.21	Pressures profile at middle area around circumference of various wave shapes liner under load condition of 252 N and shaft speed of 1260 RPM	113

4.22	Pressures profile at middle area around circumference of	
	various wave shapes liner under load condition of 441 N	
	and shaft speed of 1260 RPM	114
4.23	Peak of maximum pressure of plain liner in various shaft	
	speeds under load condition of (a) 31.5 N, (b) 63 N,	
	(c) 126 N, (d) 252 N, and (e) 441 N	117
4.24	Peak of maximum pressure of various wave shapes liner	
	in various shaft speeds under load condition of (a) 31.5 N,	
	(b) 63 N, (c) 126 N, (d) 252 N, and (e) 441 N	118
4.25	Comparison of pressure distribution numerically and	
	experimentally in the middle area circumferentially of	
	bearing with load of 31.5 N, 441 N and speed of 1260 RPM.	119
4.26	Comparison of pressure distribution numerically and	
	experimentally in the minimum film thickness area along	
	the length of bearing with load of 31.5 N, 441 N and	
	speed of 1260 RPM	120
4.26	Comparison of pressure distribution numerically and	
	experimentally in the middle area circumferentially of	
	bearing with load of 31.5 N, 441 N and speed of 2880 RPM	120
4.28	Comparison of pressure distribution numerically and	
	experimentally in the minimum film thickness area along	
	the length of bearing with load of 31.5 N, 441 N and	
	speed of 2880 RPM	121
4.29	Profile of pressure in the middle area of the bearing	
	with varies of oil supply pressures	123
4.30	Contours of pressure projected on throughout plain	

	liner bearing under lower load condition of 31.5 N and	
	shaft speed of 1260 RPM with (a) palm olein and	
	(b) engine oil as lubricant	124
4.31	Contours of pressure projected on throughout plain	
	liner bearing under load condition of 31.5 N and shaft	
	speed of 2880 RPM with (a) palm olein and	
	(b) engine oil as lubricant	125
4.32	Contours of pressure projected on throughout plain	
	liner bearing surface under load condition of 441 N and	
	shaft speed of 1260 RPM with lubricant of (a) palm olein	
	and (b) with engine oil	126
4.33	Contours of pressure projected on throughout square	
	wave liner bearing with amplitude of 600 µm under	
	load condition of 31.5 N and shaft speed of 1260 RPM,	
	(a) two square-waves liner with palm olein,(b) two square	
	waves liner with engine oil, (c) four square-waves	
	liner with palm olein, and (d) four square-waves liner	
	with engine oil as lubricant	127
4.34	Contours of pressure projected on throughout semi-circular	
	wave liner bearing under load condition of 31.5 N and shaft	
	speed of 1260 RPM, (a) 200 µm in amplitude with palm	
	olein, (b) 200 μ m amplitide with engine oil, (c) 600 μ m	
	in amplitude with palm olein, and (d) 600 µm	
	in amplitude with engine oil as lubricant	128
4.35	Contours of pressure projected on throughout sinusoidal	
	wave liner bearing with amplitude of 600 µm under load	
	condition of 31.5 N and shaft speed of 2160 RPM with	
	(a) palm olein, and (b) engine oil as lubricant	129

4.36	Comparisons of pressure profile for two square waves	
	liner bearing with amplitude of 600 µm under lower	
	load of 31.5 N and shaft speed 1260 RPM	130
4.37	Comparisons of pressure profile for two square waves	
	liner bearing with amplitude of 600 µm under lower	
	load of 31.5 N and shaft speed 2160 RPM	131
4.38	Comparisons of pressure profile for two square waves	
	liner bearing with amplitude of 600 µm under lower	
	load of 31.5 N and shaft speed 2880 RPM	132
4.39	Comparisons of pressure profile for two square waves	
	liner bearing with amplitude of 600 µm under higher	
	load of 441 N and shaft speed 1260 RPM	132
4.40	Comparisons of pressure profile for two square waves	
	liner bearing with amplitude of 600 µm under higher	
	load of 441 N and shaft speed 2160 RPM	133
4.41	Comparisons of pressure profile for two square waves	
	liner bearing with amplitude of 600 µm under higher	
	load of 441 N and shaft speed 2880 RPM	133
4.42	Comparisons of pressure profile for four square waves	
	liner bearing with amplitude of 600 µm under lower	
	load of 31.5 N and shaft speed 1260 RPM	134
4.43	Comparisons of pressure profile for four square waves	
	liner bearing with amplitude of 600 µm under lower	
	load of 31.5 N and shaft speed 2160 RPM	134
4.44	Comparisons of pressure profile for four square waves	
	liner bearing with amplitude of 600 µm under lower	

		xxii
	load of 31.5 N and shaft speed 2880 RPM	135
4.45	Comparisons of pressure profile for four waves square	
	liner bearing with amplitude of 600 µm under higher	
	load of 441 N and shaft speed 1260 RPM	136
4.46	Comparisons of pressure profile for four waves square	
	liner bearing with amplitude of 600 µm under higher	
	load of 441 N and shaft speed 2160 RPM	136
4.47	Comparisons of pressure profile for four waves square	
	liner bearing with amplitude of 600 µm under higher	
	load of 441 N and shaft speed 2880 RPM	137
4.48	Comparisons of pressure profile for semi circular wave	
	liner bearing with amplitude of 200 µm under lower	
	load of 31.5 N and shaft speed 1260 RPM	138
4.49	Comparisons of pressure profile for semi circular wave	
	liner bearing with amplitude of 200 µm under lower	
	load of 31.5 N and shaft speed 2160 RPM	138
4.50	Comparisons of pressure profile for semi circular wave	
	liner bearing with amplitude of 200 µm under lower	
	load of 31.5 N and shaft speed 2880 RPM	139
4.51	Comparisons of pressure profile for semi circular wave	
	liner bearing with amplitude of 200 µm under higher	
	load of 441 N and shaft speed 1260 RPM	139
4.52	Comparisons of pressure profile for semi circular wave	
	liner bearing with amplitude of 200 µm under higher	
	load of 441 N and shaft speed 2160 RPM	140

	•	•	•
37 37	4	1	1
- X X			
7 1 7 1	-	•	-

4.53	Comparisons of pressure profile for semi circular wave liner bearing with amplitude of 200 µm under higher	
	load of 441 N and shaft speed 2880 RPM	140
4.54	Comparisons of pressure profile for semi circular wave	
	liner bearing with amplitude of 400 µm under lower	
	load of 31.5 N and shaft speed 1260 RPM	141
4.55	Comparisons of pressure profile for semi circular wave	
	liner bearing with amplitude of 400 µm under lower	
	load of 31.5 N and shaft speed 2160 RPM	141
4.56	Comparisons of pressure profile for semi circular wave	
	liner bearing with amplitude of 400 µm under lower	
	load of 31.5 N and shaft speed 2880 RPM	142
4.57	Comparisons of pressure profile for semi circular wave	
	liner bearing with amplitude of 400 µm under higher	
	load of 441 N and shaft speed 1260 RPM	142
4.58	Comparisons of pressure profile for semi circular wave	
	liner bearing with amplitude of 400 µm under higher	
	load of 441 N and shaft speed 2160 RPM	143
4.59	Comparisons of pressure profile for semi circular wave	
	liner bearing with amplitude of 400 µm under higher	
	load of 441 N and shaft speed 2880 RPM	143
4.60	Comparisons of pressure profile for semi circular wave	
	liner bearing with amplitude of 600 µm under lower	
	load of 31.5 N and shaft speed 1260 RPM	144
4.61	Comparisons of pressure profile for semi circular wave	
	liner bearing with amplitude of 600 µm under lower	

		xxiv
	load of 31.5 N and shaft speed 2160 RPM	144
4.62	Comparisons of pressure profile for semi circular wave	
	liner bearing with amplitude of 600 µm under lower	
	load of 31.5 N and shaft speed 2880 RPM	145
4.63	Comparisons of pressure profile for semi circular wave	
	liner bearing with amplitude of 600 µm under higher	
	load of 441 N and shaft speed 1260 RPM	145
4.64	Comparisons of pressure profile for semi circular wave	
	liner bearing with amplitude of 600 µm under higher	
	load of 441 N and shaft speed 2160 RPM	146
4.65	Comparisons of pressure profile for semi circular wave	
	liner bearing with amplitude of 600 µm under higher	
	load of 441 N and shaft speed 2880 RPM	146
4.66	Comparisons of pressure profile for sinusoidal wave	
	liner bearing with amplitude of 600 µm under lower	
	load of 31.5 N and shaft speed 1260 RPM	147
4.67	Comparisons of pressure profile for sinusoidal wave	
	liner bearing with amplitude of 600 µm under lower	
	load of 31.5 N and shaft speed 2160 RPM	147
4.68	Comparisons of pressure profile for sinusoidal wave	
	liner bearing with amplitude of 600 µm under lower	
	load of 31.5 N and shaft speed 2880 RPM	148
4.69	Comparisons of pressure profile for sinusoidal wave	
	liner bearing with amplitude of 600 µm under higher	
	load of 441 N and shaft speed 1260 RPM	148

4.70	Comparisons of pressure profile for sinusoidal wave	
	liner bearing with amplitude of 600 µm under higher	
	load of 441 N and shaft speed 2160 RPM	149
4.71	Comparisons of pressure profile for sinusoidal wave	
	liner bearing with amplitude of 600 µm under higher	
	load of 441 N and shaft speed 2880 RPM	149
4.72	The highest achievement of side flow rate in various	
	cases liners bearing	162
4.73	Ratio of side flow rate to load addition in various	
	cases liners bearing	163
4.74	The highest achievement of temperature changes in	
	various cases liners bearing	164
4.75	Ratio of the temperature changes to the loads addition in	
	various cases liners bearing	164
4.76	The highest achievement of eccentricity ratio in	
	various wave liners bearing	165
4.77	The peak of maximum pressure in various waves	
	liners bearing	166
A.1	Pressure distribution circumferentially of plain liner bearing	
	in various loads lubricated by palm oil and shaft	
	speed of 1260 RPM	181
A.2	Pressure distribution circumferentially of plain liner bearing	
	in various loads lubricated by palm oil and shaft	
	speed of 2160 RPM	182

XX	V1

A.3	Pressure distribution circumferentially of plain liner bearing in various loads lubricated by palm oil and shaft	
	speed of 2800 RPM	182
A.4	Pressure distribution circumferentially of plain liner bearing	
	in various loads lubricated by engine oil and shaft	
	speed of 1260 RPM	183
A.5	Pressure distribution circumferentially of plain liner bearing	
	in various loads lubricated by engine oil and shaft	
	speed of 2160 RPM	183
A.6	Pressure distribution circumferentially of plain liner bearing	
	in various loads lubricated by engine oil and shaft	
	speed of 2800 RPM	184
A.7	Pressure distribution circumferentially of semi circular	
	wave liner bearing 200 µm amplitude in various loads	
	lubricated by palm oil and shaft speed of 1260 RPM	184
A.8	Pressure distribution circumferentially of semi circular wave	
	liner bearing 200 µm amplitude in various loads lubricated	
	by palm oil and shaft speed of 2160 RPM	185
A.9	Pressure distribution circumferentially of semi circular wave	
	liner bearing 200 µm amplitude in various loads lubricated	
	by palm oil and shaft speed of 2800 RPM	185
A.10	Pressure distribution circumferentially of semi circular wave	
	liner bearing 400 µm amplitude in various loads lubricated	
	by palm oil and shaft speed of 1260 RPM	186
A.11	Pressure distribution circumferentially of semi circular wave	
	liner bearing 400 µm amplitude in various loads lubricated	

		xxvii
	by palm oil and shaft speed of 2160 RPM	186
A.12	Pressure distribution circumferentially of semi circular wave	
	liner bearing 400 µm amplitude in various loads lubricated	
	by palm oil and shaft speed of 2800 RPM	187
A.13	Pressure distribution circumferentially of semi circular wave	
	liner bearing 600 µm amplitude in various loads lubricated	
	by palm oil and shaft speed of 1260 RPM	187
A.14	Pressure distribution circumferentially of semi circular wave	
	liner bearing 600 µm amplitude in various loads lubricated	
	by palm oil and shaft speed of 2160 RPM	188
A.15	Pressure distribution circumferentially of semi circular wave	
	liner bearing 600 µm amplitude in various loads lubricated	
	by palm oil and shaft speed of 2800 RPM	188
A.16	Pressure distribution circumferentially of semi circular wave	
	liner bearing 200 µm amplitude in various loads lubricated	
	by engine oil and shaft speed of 1260 RPM	189
A.17	Pressure distribution circumferentially of semi circular wave	
	liner bearing 200 µm amplitude in various loads lubricated	
	by engine oil and shaft speed of 2160 RPM	189
A.18	Pressure distribution circumferentially of semi circular wave	
	liner bearing 200 µm amplitude in various loads lubricated	
	by engine oil and shaft speed of 2800 RPM	190
A.19	Pressure distribution circumferentially of semi circular wave	
	liner bearing 400 µm amplitude in various loads lubricated	

by engine oil and shaft speed of 1260 RPM

190

A.20	Pressure distribution circumferentially of semi circular wave	
	liner bearing 400 µm amplitude in various loads lubricated	
	by engine oil and shaft speed of 2160 RPM	191
A.21	Pressure distribution circumferentially of semi circular wave	
	liner bearing 400 µm amplitude in various loads lubricated	
	by engine oil and shaft speed of 2800 RPM	191
A.22	Pressure distribution circumferentially of semi circular wave	
	liner bearing 600 µm amplitude in various loads lubricated	
	by engine oil and shaft speed of 1260 RPM	192
A.23	Pressure distribution circumferentially of semi circular wave	
	liner bearing 600 µm amplitude in various loads lubricated	
	by engine oil and shaft speed of 2160 RPM	192
A.24	Pressure distribution circumferentially of semi circular wave	
	liner bearing 600 µm amplitude in various loads lubricated	
	by engine oil and shaft speed of 2800 RPM	193
A.25	Pressure distribution circumferentially of two-square wave	
	liner bearing in various loads lubricated by palm oil and	
	shaft speed of 1260 RPM	193
A.26	Pressure distribution circumferentially of two-square wave	
	liner bearing in various loads lubricated by palm oil and	
	shaft speed of 2160 RPM	194
A.27	Pressure distribution circumferentially of two-square wave	
	liner bearing in various loads lubricated by palm oil and	
	shaft speed of 2800 RPM	194
A.28	Pressure distribution circumferentially of four-square wave	
	liner bearing in various loads lubricated by palm oil and	

		xxix
	shaft speed of 1260 RPM	195
A.29	Pressure distribution circumferentially of four-square wave	
	liner bearing in various loads lubricated by palm oil and shaft speed of 2160 RPM	195
A.30	Pressure distribution circumferentially of four-square wave	
	liner bearing in various loads lubricated by palm oil and shaft speed of 2800 RPM	196
A.31	Pressure distribution circumferentially of two-square wave	
	liner bearing in various loads lubricated by engine oil and shaft speed of 1260 RPM	196
A.32	Pressure distribution circumferentially of two-square wave	
	liner bearing in various loads lubricated by engine oil and shaft speed of 2160 RPM	197
A.33	Pressure distribution circumferentially of two-square wave	
	liner bearing in various loads lubricated by engine oil and shaft speed of 2800 RPM	197
A.34	Pressure distribution circumferentially of four-square wave	
	liner bearing in various loads lubricated by engine oil and shaft speed of 1260 RPM	198
A.35	Pressure distribution circumferentially of four-square wave	
	liner bearing in various loads lubricated by engine oil and shaft speed of 2160 RPM	198
A.36	Pressure distribution circumferentially of four-square wave	
	liner bearing in various loads lubricated by engine oil and shaft speed of 2800 RPM	199

A.37	Pressure distribution circumferentially of sinusoidal wave	
	liner bearing in various loads lubricated by palm oil and	
	shaft speed of 1260 RPM	199
A.38	Pressure distribution circumferentially of sinusoidal wave	
	liner bearing in various loads lubricated by palm oil and	
	shaft speed of 2160 RPM	200
A.39	Pressure distribution circumferentially of sinusoidal wave	
	liner bearing in various loads lubricated by palm oil and	
	shaft speed of 2800 RPM	200
A.40	Pressure distribution circumferentially of sinusoidal wave	
	liner bearing in various loads lubricated by engine oil and	
	shaft speed of 1260 RPM	201
A.41	Pressure distribution circumferentially of sinusoidal wave	
	liner bearing in various loads lubricated by engine oil and	
	shaft speed of 2160 RPM	201
A.42	Pressure distribution circumferentially of sinusoidal wave	
	liner bearing in various loads lubricated by engine oil and	
	shaft speed of 2800 RPM	202

LIST OF SYMBOLS

A - Wave amplitude (m)

c - Diametrical clearance of journal bearing

D - Diameter of bearing (m)

*F*_i - External body forces (N)

h - Fluid film thickness (m)

k - Consistency index

L - Length of bearing (m)

N - Shaft rotational speed (RPM)

n - Power law index

P - Static pressure (Pa)

R - Radius of circle to draw a wave circle (m)

Re - Reynolds number

S - Sommerfeld number

S_m - Mass added to the continuous phase

 T_o Reference temperature (K)

U - Velocity of shaft surface (m/s)

v - Kinematics viscosity of lubricant (m²/s)

W - Load (N)

 L_G - Length of a half wave (m)

 n_V - Sum of wave valley

 n_C - Sum of wave crest

 Q_x - Lubricant flow rate (m³/s)

 Q_{SC} - Lubricant flow rate for semi circular wave liner (m³/s)

 Q_{SQ} - Lubricant flow rate for square wave liner (m³/s)

 Q_{Sin} - Lubricant flow rate for sinusoidal wave liner (m³/s)

Qinlet - Sum of total oil enter the bearing (m³/s)

Qoutlet - Sum of total oil out the bearing (m^3/s)

Qends - Sum of oil collected from both of the ends of bearing (m³/s)

Qloses - Sum of lubricant fluid uncollected (m³/s)

 R_b - Radius of bearing (m)

 r_j Radius of shaft (m)

Angle of radius to draw a wave circle (°)

e - Eccentricity (m)

V - Eccentricity ratio, $\frac{e}{c}$

Shear rate (1/s)

 μ - Absolute viscosity (N.s/ m²)

LIST OF APPENDICES

APPENDIX	TITLE	PAGE
A	Pressure Distribution in Various Loads and Shaft Speeds	
	Circumferentially of Journal Bearings	181
В	List of Publications	203

CHAPTER 1

INTRODUCTION

In this study, current journal bearings designed are proposed to be improved by modifications of the inner side of the bearing at the usual location of the shaft slide. Performance of journal bearing has been reported to be greatly affected by changes in the profile of inner side surface (liner). This change has great effect on pressure distribution, lubricant temperature reduction, and side flow rate increment. All mentioned performance measurements are analyzed and compared to normal journal bearings.

1.1 Background of the Problem

Hydrodynamic lubrication is a condition where journal bearings operate in thick film creating sufficient hydrodynamic pressure to support the load (Bhusan, 2002). When fluid film is thick enough then the hydrodynamic lubrication condition can ensure solid contact does not occur. In many years, efforts deliver lubricant into the bearing has been considered. The aim is to ensure sufficient lubricant is always present in the journal bearing. In an attempt to meets this aspect, geometries of liner

bearings has been modified. Flack *et al.* (1980) introduced four lobes journal bearing and show that the maximum pressure occurred at middle area, circumferentially. Similarly, Goyal and Sinhasan (1991) developed two lobes journal bearing. From their study, it was shown that as the load increases the minimum film thickness reduces. The side flow rate also increases as the load increases. These efforts result in higher stability but lower in load carrying capacity. To complement the above findings, Dimofte (1995) conducted a study by developing three wave liner journal bearing with ratio of wave amplitude to radial clearance was 0.2 and 0.4, as in Figure 1.1. The result shows an increase in load carrying capacity.

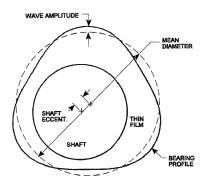


Figure 1.1: Three wave liner journal bearing, Dimofte (1995)

Groove location has also been considered in the study of journal bearing. As reported by Costa *et al.* (2000) who has developed a steady load journal bearing in order to investigate the effect of three different axial groove location mounted on the bearing. They reported that the changes in groove locations have significantly affect the pressure characteristic in journal bearing. Moreover, other researchers also investigated the journal bearing with journal groove shape assorted to achieve a good performance, as mentioned by Sahu *et al.* (2006) and Hirayama *et al.* (2009). They found that the modification cause higher eccentricity.

The thermal effect on the journal bearing performance has also been studied. Van Ostayen and Van Beek (2009) have investigated the thermal effect on lemon bore liner journal bearing. They concluded that the result, in the condition of various shaft speeds and load constant, have shown that the maximum temperature remains constant. Also, the maximum temperatures are affected significantly by changed in type and viscosity of lubricant. As the theoretical study has been carried out by Ene

et al. (2007) on a wave journal bearing was shown that maximum temperature of the bearing take place in the vicinity of minimum fluid films thickness then decreased in region of maximum fluid films thickness. In addition, increases in the load experienced by the bearing also cause a raise in temperature distribution. Furthermore, Chauhan et al. (2010) has carried out the study concerning thermohydrodynamics analysis on the elliptical journal bearing (see Figure 1.2) with different grades lubricant. They found that as the shaft speed and eccentricity ratio increases, it affects the film temperature and thermal pressure.

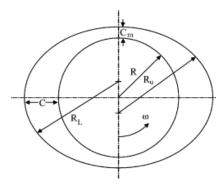


Figure 1.2: Elliptical journal bearing, Chauhan *et al.* (2010)

Various types of journal bearings have an important role in assisting the machine work although the data of about the characteristics and performance it under various operating conditions are lack and not abundantly available. Although numerous studies done in various methods that involve numerical analysis and experimental investigations. Based on the previously mentioned findings, generally the studies are focused on the liner-shape and groove arrangement on journal bearing. The combination of wave and groove on surface of liner bearing are a gap that would be adopted as topic in the present study associated with journal bearing.

From the previous studies, modifications of the liner have been show to increase the load carrying capacity of the journal bearing. However other problem could occur if modifications are not done correctly. This study introduces several new types of liner bearings and performance of the bearing has been evaluated. The effect of other lubricant, in order to study the effect of different lubricant properties, is also included.

1.2 Problem Statement

The modifications on the liner bearing have been studied and produce the improvement on journal bearing performance. The performance of journal bearing with liner modifications is very dependent on the operating conditions as well. There are variety operating conditions that might encounter in practice. Temperature operation, load imposed, shaft speed, lubricant condition are among other that have many effects on journal bearing performance. Either the parameter that most influence on the journal bearing operation was the change in the shape of the liner. Its changes to give effect to the pressure distribution, oil flow rate, load carrying capacity and journal bearing temperature. It was identified there is some profile of liner surface that have not been investigated. The changes of liner shape may affect on heat generated, pressure distribution, and lubricant flow behavior in journal bearing, are aims of the study. New types of various shapes wave liner are introduced in order to complement the lack of data. The effect of different types of lubricants to the modification was also studied.

1.3 Objectives

The objectives of present study are, to determine the effect of new type of liner, such as square wave, semi-circular wave and sinusoidal wave on:

- i.) The side flow rate of journal bearing
- ii.) The temperature change of the lubricant journal bearing
- iii.) The eccentricity of journal bearing
- iv.) The pressure distribution of journal bearing

- v.) Comparisons of wave liner bearing performance to a plain liner journal bearing.
- vi.) Pressure distribution of wave liner journal bearing experimentally and numerically aims to comparison.

1.4 Scope of Research

The scopes of this study directed as follows:

- 1. Numerical and experimental approach have been used for comparison
- 2. Mineral oil has been used as reference (SAE 20W-40)
- 3. One type of other lubricant, that is palm olein, has been analyzed.
- 4. Specific range of speed has been applied (1000 RPM -3000 RPM)
- 5. Load increment is limited up to 450 N to avoid metal contact.
- 6. Ambient pressure and temperature standard operating conditions where the experiment is carried out in laboratory.
- 7. The wave liner bearings (semi circular wave, square wave, sinusoidal wave) compared to a plain liner journal bearing.
- 8. The gravity oil supply system has been applied to reduce operational effect.

1.5 Research Contributions

- 1. This study provides the completeness of data to journal bearing with the effect of wavy liner surface for various wave shapes.
- 2. The data of this study can be used to support the design of journal bearing.
- 3. The non-mineral oil performs analysis can be used as feasibility data of lubricant for journal bearing applications.
- 4. This study will supplement the similarity and universality aspects of deficit lubricant behavior on type of wave liner bearing, as already known in the narrow gap, so then a practical characterization of the gap is complete possible.

1.6 Outline of Thesis

As an introduction to the whole of thesis the background of problem, problem statement, objectives, scope of research, and research contribution were outlined in this chapter.

Literature reviews about of the studies of journal bearing were outlined in Chapter 2. The chapter initiated with the review in relation to main component that was found in the journal bearing. Then followed by reconsider on some studies that are possible to support the use of bio-based as lubricant. To obtain the guidelines in the discussion of the results should be reviewed several studies, among others, about the thermal characteristics and pressure distribution. Subsequent to examine the extent to which the development of liner bearing deformation in recent years are summarized in a study of non-circular liner bearing and grooved journal bearing. At

the end of this chapter closes with an overview of research methods such as numerical study in the journal bearing.

Chapter 3 are the pages to describe the research methodology used during the study. In it contains the description of the lubricants, experimental setup, wavy liner bearing concepts, fluid film pressure measurement, fluid film thickness measurement, oil side flow measurement and data processing. The numerical by using Fluent CFD components were reviewed and then the validation has been conducted.

Results and discussion on the act of journal bearing were presented in Chapter 4. Some achievements were realized in the form of oil side flow, oil temperature, eccentricity, and pressure distribution in journal bearing.

Chapter 5 is to be the last place to express the essence of this study. This chapter contains the conclusion and recommendations in the future work.

REFERENCES

- Bouard, L, Fillon, M, and Frene, J. (1996). Comparison between three turbulent models-application to therhydrodynamic performances of tilting-pad journal bearing. *Tribology International*, Vol 29, No 1, 11-18.
- Bhusan, Bharat. (2002). Introduction To Tribology. Jhon Wiley & Sons, Inc.
- Bayrakçeken, H and Yürüsoy, M. (2006). Comparison of pressure distribution in inclined and parabolic slider bearing. *Mathematical and Computational Applications*, Vol 11, No.1, pp 65-73.
- Brito, F.P, Miranda, A.S, Bouyer, J, and Fillon, M. (2006). Experimental investigation of the influence of supply temperature and supply pressure on the performance of a two axial hydrodynamic journal bearing. *Proceeding of IJT: STLE/ASME International Joint Tribology Conference*, October 23-25, san Antonio, TX, USA.
- Buzescu, Daniela.M, and Pascovici, Mircea D. (2007). Seizure of high speed three wave journal bearing. *10th International Conference On Tribology*. November 8-10, Bucharest, Romania.
- Costa, L, Fillon, M, Miranda, A.S, Claro, J.C.P. (2000). An experimental investigation of the effect of groove location and supply pressure on the THD performance of a steadily loaded journal bearing. *Journal of Tribology*, 122, 227.

- Cioc, Sorin, Dimofte, Florin, and Keith, Theo.G.Jr. (2003). Application of the CE/SE method to wave journal bearings. *Tribology Transaction*, 46, 2, 179-186.
- Chiang, Hsiu-Lu, Hsu, Cheng-Hsing, and Lin, Jaw-Ren. (2004). Lubrication performance of finite journal bearings considering effects of couple stresses and surface roughness. *Tribology International*, 37,297-307.
- Cheenkachorn, Kraipat, and Fungtammasan, Bundit. (2010). Development of engine oil using palm oil as a base stock for four-stroke engines. *Energy*, 35, 2552-2556.
- Chauhan, Amit, Sehgal, Rakesh, and Sharma, Rajesh Kumar. (2010). Thermohydrodynamic analysis of elliptical journal bearing with different grade oils. *Tribology International*, 43, 1970-1977.
- Chauhan, Amit, Sehgal, Rakesh, and Sharma, Rajesh Kumar. (2011). Investigation on the thermal effects in non-circular journal bearings. *Tribology International*, 44, 1765-1773.
- Dimofte, Florin. (1995). Wave journal bearing with compressible lubricant-part I: The wave bearing concept and a comparison to the circular bearing. *Tribology Transactions*, 38,1,153-160.
- Dimofte, Florin. (1995). Wave journal bearing with compressible lubricant-part II: A comparison of the wave bearing with a wave-groove bearing and a lobe bearing. *Tribology Transactions*, 38, 2, 364-372.
- Din, Massimo Del, and Kassfeldt, Elisabet. (1999). Wear characteristics with mixed lubrication conditions in a full scale journal bearing. *Wear*, 232, 192-198.
- Dimofte, florin, and Hendricks, Robert.C. (2002). Wave journal bearings under dynamic loads. *NASA/TM*-211079.

- Dimofte, Florin, Fleming, David.P, Anderson, William.J, and Klein, Richard.C. (2005). Test of a fluid-film wave bearing at 350° C with liquid lubricants. *Tribology Transaction*, 48,4.
- Dimofte, Florin, Ene, Nicoleta. M, Handschuh, Robert.F and Keith Theo. G. (2007). New rig test journal oil lubricated wave bearing. *Proceeding of ASME/STLE international Joint Tribology Conference*, October 22-24, San Diego, California USA.
- Ene, Nicoleta. M, Dimofte, Florin, Keith, Theo. G, and Handschuh, Robert.F. (2007). Temperature distribution of a wave journal bearing comparison with test data. *Proceeding of ASME/STLE international Joint Tribology Conference*, October 22-24, San Diego, California USA.
- Ene, Nicoleta M., Dimofte, Florin, Keith, Theo G. Jr. (2008). A stability analysis for a hydrodynamic three-wave journal bearing. *Tribology International*, 41,434-442.
- Ene, Nicoleta M., Dimofte, Florin, Keith, Theo G. Jr. (2008). A Dynamic analysis of hydrodynamic wave journal bearings. *Tribology Transaction*, 51, 82-91.
- Erhan, Sevim Z, Sharma, Brajendra K, and Perez, Joseph M. (2006). Oxidation and low temperature stability of vegetable oil-based lubricants. *Industrial Crops and Products An International Journal*, 24, 292-299.
- Flack, R.D, Leader, M.E, and Allaire, P.E. (1980). An experimental and theoretical investigation of pressures in four lobe bearings. *Wear*, 61,233-242.
- Fox, N.J, Stachowiak, G.W. (2007). Vegetable oil based-lubricants- A review of oxidation. *Tribology International*, 40, 1035-1046.
- Goyal, K.C and Sinhasan, R. (1991). Elastohydrodynamic studies of two-lobe journal bearings with non-Newtonian lubricants. *Wear*, 145, 329-345.

- Gawrilow, Ilija. (2003). Palm oil usage in lubricants. 3rd Global and Fats Business Forum, October 8, USA.
- Gertzos, K.p, Nikolakopoulos, P.G, and Papadopoulos, C.A. (2008). CFD analysis of journal bearing hydrodynamic lubrication by Bingham lubricant. *Tribology International*, 41, 1190-1204.
- Hargreaves, D.J. (1991). Surface waviness effects on the load-carrying capacity of rectangular slider bearings. *Wear*, 137-151.
- Hamrock, Bernard.J. (1994). Fundamentals of fluid film lubrication. McGraw-Hill Series in Mechanical Engineering.
- Horng, Jeng Haur. (1998). Studies of tribological behavior and separation between surfaces at initial boundary lubrication. *Wear*, 216, 8-14.
- Hassan, A.B, Abolarin, M.S, Nasir, A, and ratchel, U. (2006). Investigation on the use of palm olein as lubrication oil. *Leonardo electronic journal of Practices and Technologies*, 8, 1-8.
- Hirayama, Tomoko, Yamaguchi, Naomi, Sakai, Shingo, Hishida, Noriaki, Matsuoka, Takashi, and Yabe, Hiroshi. (2009). Optimization of groove dimensions in herringbone-grooved journal bearings for improved repeatable run-out characteristics. *Tribology international*, 42, 675-681.
- Habchi, W, Vergne, P, Bair, S, Anderson, O, Eyheramendy, D, and Morales-Espejel, G.E. (2010). Influence of pressure and temperature dependence of thermal properties of a lubricant on the behavior of circular TEHD contacts. *Tribology International*, 43,1842-1850.
- Ju, Sheu-Ming, and Weng, Cheng-I. (1994). Thermohydrodynamic analysis of finite-width journal bearing with non-newtonian lubricants. *Wear*, 171,41-49.

- Jang, J.Y, and Khonsari, M.M. (2002). Thermohydrodynamic analysis of journal bearings lubricated with multigrade oils. *CMES*, 3, 4, 455-464.
- Kumar, Punit, Jain, S.C, and Ray, S. (2001). Study of surface roughness effects in elastohydrodynamic lubrication of rolling line contacts using a deterministic model. *Tribology International*, 34, 713-722.
- Khonsary, Michael. M, and Booser, E. Richard. (2008). *Applied Tribology Bearing Design and Lubrication*. (2nd Edition). Chichester, England: Jhon Willey and Sons, Ltd.
- Shigley, Joseph.E, Mischke, Charles.R, and Budynas, Richard. G. (2004). *Mechanical engineering design*. (Seventh Edition). New York: McGraw-Hill
- Lingard, S, Chen, N.N.S, and Kong, Y.C. (1982). Aspects of the performance of externally pressurized journal bearings. *Wear*, 78, 343-353.
- Li, X.K, Gwynllyw, D.Rh, Davies, A.R, and Philips, T.N. (2000). On the influence of lubricant properties on the dynamics of two-dimensional journal bearings. *Journal Non-Newtonian Fluid Mechanic*, 93, 29-59.
- Landsdown, A.R. (2004). *Lubrication and lubricant selection a practical guide*. third edition. Profesional Engineering Publishing Limited, London and Bury St Edmunds, UK.
- Larsson, R. (2009). Modelling the effect of surface roughness on lubrication in all regimes. *Tribology International*, 42, 512-516.
- Majumdar, B.C, and Saha, A.K. (1974). Temperature distribution in oil journal bearing. *Wear*, 28, 259-266.
- Mitsui, J, Hori, Y, and Tanaka, M. (1983). Thermohydrodinamic analysis of cooling effect of supply oil in circular journal bearing. *Transaction By ASME*,414/ Vol. 105.

- Ma, M.T and Taylor, C.M. (1995). Effects of oil feed temperature on the performance of an elliptical bore bearing. *Lubricants and Lubrication*, Elsevier Science.
- Masjuki, H.H, Sapuan, S.M. (1995). Palm oil methyl ester as lubricant additive in a small diesel engine. *JAOCS*, 72, No.5.
- Ma, M.T, and Taylor, C.M. (1996). An experimental investigation of thermal effects in circular and elliptical plain journal bearings. *Tribology International*, 29, 19-26.
- Masjuki, H.H, Maleque, M.A. (1997). Investigation of the anti-wear characteristics of palm oil methyl ester using a four-ball tribometer test. *Wear*, 206,179-186.
- Masjuki, H.H, Maleque, M.A, Kubo, A, Nonaka, T. (1999). Palm oil and mineral based lubricants-their tribological and emission performance. *Tribology International*, 32,305-314.
- Maleque, M.A, Masjuki, H.H, and Haseeb, A.S.M.A. (2000). Effect of mechanical factors on tribological properties of palm oil methyl ester blended lubricant. *Wear*, 239, 117-125.
- Mott, Robert, L. (2004). *Machine elements in mechanical design*. fourth edition. pearson prentice hall.
- Mishra, P.C, Pandey, R.K, and Athre, K. (2007). Temperature profile of an elliptic bore journal bearing. *Tribology International*, 40, 453-458.
- Mongkolwongrojn, Mongkol, Limkul, Matee, and Kuanoon, Kunchit. (2007). Theoritical characteristics of journal bearings lubricated with non-newtonian lubricants included thermal effects. *The 21st Conference of Mechanical Engineering Network of Thailand*, 17-19 October, Chonburi, Thailand.

- Muhsen, Ali Ibrahem, and Al-Amer, Ali.A. (2007). Comparative study between Newtonian and non-newtonian lubricants in journal bearing using variable viscosity. *Engineering and Technology*. 25, 6.
- Ma, Yan-Yan. (2008). Performance of dynamically loaded journal bearings lubricated with couple stress fluids considering the elasticity of the liner. *Journal of Zheijiang University SCIENCE A*, 9(7), 916-921.
- Mehta, N.P, Rattan, S.S, and Verma, Rajiv (2010). Stability analysis of two lobe hydrodynamic journal bearing with couple stress lubricant. *ARPN Journal of Engineering and Applied Sciences*, 5,1.
- Molka, Attia Hili, Slim, Bouaziz, Mohamed, Maatar, Tahar, Fakhfakh, and Mohamed, Haddar. (2010). Hydrodynamic and elastohydrodynamic studies of a cylindrical journal bearing. *Journal of Hydrodynamic*, 22 (2), 155-163.
- Mongkolwongrojn, Mongkol and Aiumpronsin, Chatchai. (2010). Stability analysis of rough journal bearings under TEHL with non-Newtonian lubricants. *Tribology International*, 43, 1027-1034.
- Naduvinamani, N.B, Fathima, Syeda Tasneem, and Hiremath, P.S. (2003). Hydrodynamic lubrication of rough slider bearings with couple stress fluids. *Tribology International*, 36,949-959
- Nuruzuman, D.M, Khalil, M.K, Chowdhury, M.A and Rahaman, M.L. (2010). Study on pressure distribution and load capacity of a journal bearing using finite element method and analytical method. *International Journal of Mechanical & Mechatronics Engineering IJME-IJENS*, 10, 5.
- Pai, R, Hargreaves, D.J, and Brown, R. (2001). Modelling of fluid flow in a 3-axial groove water bearing using computational fluid dynamics. *14th Australasian Fluid Mechanics Conference Adelaide University*, 10-14 December, Adelaide, Australia.

- Quinchia, L.A, Delgado, M.A, Valencia, C, Franco, J.M, and Gallegos, C. (2010).
 Viscosity modification of different vegetable oils with EVA copolymer for lubricant applications. *Industrial Crops and Products*, 32, 607–612.
- Read, L.J, and Flack, R.D. (1987). Temperature, pressure and film thickness measurement for an offset half bearing. *Wear*, 117, 197-210.
- Rao, Ramamohana. A and Mohanram, P.V. (1994). A study of wear characteristics mixed-lubrication conditions of journal bearings operating under mixed lubrication condition. *Wear*, 172, 11-22.
- Rasheed, Hassan .E. (1998). Effect of surface waviness on the hydrodynamic lubrication of a plain cylindrical sliding element bearing. *Wear*, 223,1-6.
- Roy, L. (2009). Thermo-hydrodynamic performance of grooved oil journal bearing. *Tribology International*, 42, 1187-1198.
- Roy, L, and Laha, S.K. (2009). Steady state and dynamic characteristics of axial grooved journal bearings. *Tribology International*, 42, 754-761.
- Swamy, S.T.N, Prabhu, B.S, and Rao, B.V.A. (1975). Calcultaed load capacity of non-newtonian lubricants in finite width journal bearing. *Wear*, 31, 277-285.
- Safar, Z.S. (1979). Journal bearings operating with non-newtonian lubricant films. *Wear*, 53, 95-100.
- Sheeja, D and Prabhu, B.S. (1992). Thermohydrodynamic lubrication of non-newtonian journal bearings: theory and experiments. *Journal of . Physics D:Applied Physics*, 25,1706-1712.
- Syverud, Tor and Tanaka, Masato. (1997). Experimental investigation of the effect of shaft heating and cooling on single bore journal bearing. *Wear*, 207, 112-117.

- Sinanoglu, Cem, Nair, Fehmi, and Karamis, M. Baki. (2005). Effects of shaft surface texture on journal bearing pressure distribution. *Journal of Materials Processing Technology*, 168, 344-353.
- Stachowiak, Gwidon W, and Batchelor, Andrew W. (2005). *Engineering Tribology*. third edition. Elsevier Butterworth-Heinemann, 30 corporate drive, suite 400, Burlington, MA 01803, USA, Linacre House, Jordan Hill, Oxford OX2 8DP, UK.
- Sahu, M, Sarangi, M, and Majumdar, B.C. (2006). Thermo-hydrodynamic analysis of herringbone grooved journal bearings. *Tribology International*, 39, 1395-1404.
- Sahlin, Fredrik, Almqvist, Andreas, Larson, Roalnd, and Glavatskih, Sergei. (2007). Rough surface flow factors in full film lubrication based on a homogenization technique. *Tribology International*, 40,1025-1034.
- Singh, U, Roy, L, and Sahu, M. (2008). Steady-state thermo-hydrodynamic analysis of cylindrical fluid film journal bearing with an axial groove. *Tribology International*, 41, 1135-1144.
- Sharma, R.K and Pandey, R.K. (2009). Experimental study of pressure distribution in finite slider bearing with single continuous surface profiles on the pads. *Tribology International*, 42, 1040-1045.
- Shenoy, B.S, Pai, R.S, Rao, D.S, and Pai, R. (2009). Elasto-hydrodynamic lubrication analysis of full 360o journal bearing using CFD and FSI techniques. *World Journal of Modeling and Simulation*, vol 5, no 4, 315-320.
- Shi, Xi, and Ni, Ting. (2011). Effects of groove textures on fully lubricated sliding with cavitation. *Tribology International*, 44, 2022-2028.
- Tayal, S.P. (1981). Analysis of Hydrodynamic journal bearing with non-newtonian power law lubricants by the finite element method. *Wear*, 71, 15-27.

- Tender, K. (1996). Dynamic of rough slider bearings: effects of one-sided roughness/waviness. *Tribology International*, 29,2, 117-122.
- Tanaka, Masato. (2000). Journal bearing performance under starved lubrication. *Tribology International*, 33, 259-264.
- Tala-Ighil, Naser, Maspeyrot, Patrick, Fillon, Michel, and Bounif, Abdelhamid. (2007). Hydrodynamic effects of texture geometries on journal bearing surfaces. 10th International Conference on Tribology. November 8-10, Bucharest Romania.
- Ostayen, Ron A.J. Van, and Beek, Anton. Van. (2009). Thermal modelling of the lemon-bore hydrodynamic bearing. *Tribology International*, 42, 23-32.
- Venkateswarlu, K, Rao, N.J, Venugopal, E.V, and Akella, S. (1990). Three-dimensional laminar and turbulent lubrication in journal bearing, *Wear*, 126, 263-279.
- Vincent, Bruno, Maspeyrot, Patrick, and Frene, Jean. (1996). Cavitation in dynamically loaded journal bearings using mobility method. *Wear*, 193, 155-162.
- Walker, James.F, Dimofte, Florin and Eddy, Harold.E. (1995). Wave journal bearing; part II: Experimental pressure measurement and fractional frequency whirl threshold for wave and plain journal bearings. *NASA Lewis Research Center*.
- Willing, Andreas. (2001). Lubricants based on renewable resources-an environmentally compatible alternative to mineral oil products. *Chemosphere*, 43, 89-98.
- Wang, Yansong, Zhang, Chao, Wang, Q. Jane, and Lin, Chih. (2002). A mixed-TEHD analysis and experiment of journal bearings under severe operating conditions. *Tribology International*, 35, 395-407.

- Wu, C.W, and Ma, G.J. (2005). Abnormal behavior of a hydrodynamic lubrication journal bearing caused by wall slip. *Tribology International*, 38, 492-499.
- Wang, Wen-Zhong, Chen, Hui, Hu, Yuan-zhong, and Wang, Hui. (2006). Effect of surface roughness parameters on mixed lubrication characteristics. *Tribology International*, 39, 522-527.
- Wang, Xiao Li and Zhu, Ke-Qin. (2006). Numerical analysis of journal bearings lubricated with micropolar fluids including thermal and cavitating effects. *Tribology International*, 39, 227-237.
- Wan Nik, Wan Mohd Norsani (2006). Performance investigation of energy transport media as influenced by crop based properties. Laporan Akhir Penyelidikan. Universiti Teknologi Malaysia.
- Wang, J.K and Khonsari, M.M. (2008). Effects of oil inlet pressure and inlet position of axially grooved infinitely long journal bearings. Part I: Analytical solutions and static performance. Tribology International, 41, 119-131.
- Zhao, Hongmei, Choy, F.K, and Braun, M.J. (2002). Modeling and analysis of a wavy thrust bearing. *Tribology Transaction*, 45,1,85-93.
- Zhao, Hongmei, Choy, F.K, and Braun, M.J. (2005). Dynamic characteristics and stability analysis of a wavy thrust bearing. *Tribology Transaction*, 48,133-139.