MODELING AND VIBRATION CONTROL OF A GANTRY CRANE

ZAIRULAZHA BIN ZAINAL

A project report submitted in partial fulfillment of the requirement for the award of the degree of Master of Engineering (Electrical - Mechatronics & Automatic Control)

Faculty of Electrical Engineering

Universiti Teknologi Malaysia

MARCH, 2005

To my beloved family, mum, dad, Faizal and Ani, thank you for your support and encouragement that you have given in my life. Also to my fiancée, Saleha, thank you for understanding me and the supports that been given to me in completing this work.

ACKNOWLEDGEMENT

First and foremost, I would like to take this opportunity to thank my project supervisor, Dr. Zaharuddin Mohamed for his invaluable guidance, assistance and support throughout the project. Under his supervision, many aspects regarding on this project has been explored, and with the knowledge, idea and support received from him, this thesis can be presented in the time given.

I also would like to thank Mr. Yusmady Ariffin and Mr. Termiti Sidon, because of their willingness to be as guarantor to secure my scholarship. For all lecturers involved in teaching my course, thanks for the lesson that been delivered. Not forget to all my friends, course mate, and anyone that has provided, whether an idea or support, directly or indirectly, that played a role towards in completing this work. Unfortunately, it is not possible to list all of them in this limited space. Also special thanks to UTM for giving me an opportunity to pursue my course of study. Also for FKE, that has given not only knowledge, but also experiences during my study here.

ABSTRACT

The aim of this project is to model industrial gantry cranes for investigations of dynamic behavior of the system. A gantry crane incorporating a payload is considered. The modeling technique which is easier in terms of mathematical derivation is chosen. Comparison between the mathematical model derived and previous work is conducted to validate the dynamic model. A simulated time response and vibration frequency of the system to an input command is presented. Furthermore, a vibration control scheme based on an open-loop filtering technique is developed. Finally, the effectiveness of the controller is investigated in terms of time response, level of vibration reduction, robustness and the capability of handling a payload.

ABSTRAK

Objektif projek ini adalah untuk melaksanakan permodelan terhadap kren yang disokong oleh rangka besi dengan tujuan untuk mengkaji sifat-sifat dinamik sistem tersebut. Pertimbangan terhadap kren yang melibatkan beban akan dilaksanakan. Kaedah permodelan dipilih daripada beberapa kaedah yang telah dikenalpasti, dimana persamaan matematik bagi sistem tersebut dapat ditunjukkan dalam bentuk yang paling ringkas. Perbandingan di antara model matematik dan kerja-kerja yang telah dilaksanakan pada masa lampau dilakukan. Simulasi berkaitan dengan tindakbalas masa dan frekuensi getaran sistem terhadap masukan akan dikaji. Selain daripada itu, kawalan getaran berdasarkan pada gelung bukaan teknik penapisan akan dibangunkan. Pengawal yang telah dibangunkan akan dikaji keefektifannya dari sudut tindakbalas masa, kadar pengurangan getaran, ketegapannya dan kebolehan untuk membawa beban.

CONTENTS

1	INTRODUCTION		

1.1 Crane: Overview	1
1.2 Gantry Crane	4
1.3 Motivation, Rational, Significance and Need	
for the Study	5
1.4 Objectives and Work Methodology	6
1.5 Thesis Outline	8

2	LITERATURE REVIEW	
	2.1 Modeling and Control of Gantry Cranes	10
	2.2 Modeling of Gantry Cranes	12
	2.3 Control of Gantry Cranes	14
	2.4 Conclusion	17

3	GANTRY	CRANE
5	UMUM	

CHAPTER TITLE

PAGE

19

5.2 Assumption and Emilation

MODELING OF A GANTRY CRANE	
4.1 Introduction	21
4.2 Modeling Techniques	23
4.3 Mathematical Modeling for a Gantry Crane	29
4.4 Simulation of the Gantry Crane	39
4.4.1 Model Setup	39
4.5 Verification of the Mathematical Model	46
4.5.1 Model Verification	49
4.6 Model Characteristics	51

5 VIBRATION CONTROL FOR A GANTRY

CRANE

4

5.1 Prologue	56
5.2 Controlling the Crane via Feed-Forward Control	
Strategy	59
5.3 Command Shaping: Filter Design's Implementation	65
5.4 Comparative Performance Assessment	121

6 CONCLUSION AND RECOMMENDATION FOR

FUTURE WORK

6.1 Conclusion	133

6.2 Recommendation for Future Work	134
old Recommendation for Future Work	101

REFERENCES	136

LIST OF TABLES

TABLE NO.TITLE

5.1	System Response for Designed Filter with Exact	122
	Natural Frequency (Cart Displacement)	
5.2	System Response for Designed Filter with Exact	124
	Natural Frequency (Load Swing)	
5.3	System Response for Designed Filter with 20%	125
	Error in Exact Natural Frequency (Cart	
	Displacement)	
5.4	System Response for Designed Filter with 20%	127
	Error in Exact Natural Frequency (Load Swing)	

PAGE

LIST OF FIGURES

FIGURE NO. TITLE

PAGE

1.1	Overhead crane			
1.2	Gantry crane			
1.3	Work Methodology			
3.1	Gantry crane model			
4.1	Block diagram of linear system			
4.2	F(t) approximated by sequence of pulses			
4.3	Cart's free body diagram			
4.4	Load's free body diagram			
4.5	Block diagram based on state-space equation			
4.6	Modified block diagram	36		
4.7	System's block diagram (a) System	39		
	configuration (b) Configuration inside the crane			
	dynamics block			

4.8	Configuration for crane dynamics (a) system's		
	parameter variable (b) state-space model		
	configuration		
4.9	Input setting	41	
4.10	Settings of parameter variable		

- 4.11Cart displacement424.12Load oscillation43
- 4.13 New system's configuration 44
- 4.14Input modifications444.15Cart displacement45
- 4.16Load oscillation454.17Frequency response of the system50
- 4.18Load parameter setting (0.5 kg)52
- 4.19Load mass parameter setting (10 kg)52
- 4.20 Simulation results for load mass 0.5 kg (a) cart 53 displacement (b) load oscillation
- 4.21 Simulation results for load mass 10 kg (a) cart 54 displacement (b) load oscillation
- 5.1 Feed-forward control configuration 65
- 5.2Bang-bang force input665.25.25.2
- 5.3 Filtered input using 1st order Butterworth 67
 low-pass filter (with cut-off frequency at 25% of the exact natural frequency (1.29 rad/s))

- 5.4 Filtered input using 3rd order Butterworth 67
 low-pass filter (with cut-off frequency at 25% of the exact natural frequency (1.29 rad/s))
- 5.5 Filtered input using 1st order Butterworth 68
 low-pass filter (with cut-off frequency at 75% of the exact natural frequency (3.87 rad/s)
- 5.6 Filtered input using 3rd order Butterworth 68
 low-pass filter (with cut-off frequency at 75% of the exact natural frequency (3.87 rad/s)
- 5.7 Response of the gantry crane to unshaped input69 with exact natural frequency (a) cartdisplacement (b) load oscillation
- 5.8 Response of the gantry crane to low-pass filtered 70 input (1st order Butterworth filter with cut-off frequency at 25% of natural frequency) with exact natural frequency (a) cart displacement (b) load oscillation
- 5.9 Response of the gantry crane to low-pass filtered 71 input (3rd order Butterworth filter with cut-off frequency at 25% of natural frequency) with exact natural frequency (a) cart displacement (b) load oscillation

- 5.10 Response of the gantry crane to low-pass filtered 72 input (1st order Butterworth filter with cut-off frequency at 75% of natural frequency) with exact natural frequency (a) cart displacement (b) load oscillation
- 5.11 Response of the gantry crane to low-pass filtered 73 input (3rd order Butterworth filter with cut-off frequency at 75% of natural frequency) with exact natural frequency (a) cart displacement (b) load oscillation
- 5.12 Filtered input using 1st order elliptic low-pass 74 filter (with cut-off frequency at 25% of the exact natural frequency (1.29 rad/s))
- 5.13 Filtered input using 3rd order elliptic low-pass 75
 filter (with cut-off frequency at 25% of the exact natural frequency (1.29 rad/s))
- 5.14 Filtered input using 1st order elliptic low-pass 75 filter (with cut-off frequency at 75% of the exact natural frequency (3.87 rad/s))
- 5.15 Filtered input using 3rd order elliptic low-pass 76 filter (with cut-off frequency at 75% of the exact natural frequency (3.87 rad/s))

- 5.16 Response of the gantry crane to low-pass filtered 77 input (1st order elliptic filter with cut-off frequency at 25% of natural frequency) with exact natural frequency (a) cart displacement (b) load oscillation
- 5.17 Response of the gantry crane to low-pass filtered 78 input (3rd order elliptic filter with cut-off frequency at 25% of natural frequency) with exact natural frequency (a) cart displacement (b) load oscillation
- 5.18 Response of the gantry crane to low-pass filtered 79 input (1st order elliptic filter with cut-off frequency at 75% of natural frequency) with exact natural frequency (a) cart displacement (b) load oscillation
- 5.19 Response of the gantry crane to low-pass filtered 80 input (3rd order elliptic filter with cut-off frequency at 75% of natural frequency) with exact natural frequency (a) cart displacement (b) load oscillation
- 5.20 Magnitude response of 1st order Butterworth 82 band-stop filter
- 5.21 Magnitude response of 3rd order Butterworth 82 band-stop filter

5.22	Magnitude	response	of	1^{st}	order	elliptic	83	
	band-stop filter							

- 5.23 Magnitude response of 3rd order elliptic 84 band-stop filter
- 5.24 Filtered input using 1st order Butterworth 85 band-stop filter
- 5.25 Filtered input using 3rd order Butterworth 85 band-stop filter
- 5.26 Filtered input using 1st order elliptic band-stop 86 filter
- 5.27 Filtered input using 3rd order elliptic band-stop 86 filter
- 5.28 Response of the gantry crane to 1st order 87 Butterworth band-stop filter's filtered input with exact natural frequency (a) cart displacement (b) load oscillation
- 5.29 Response of the gantry crane to 3rd order 88 Butterworth band-stop filter's filtered input with exact natural frequency (a) cart displacement (b) load oscillation
- 5.30 Response of the gantry crane to 1st order elliptic 89
 band-stop filter's filtered input with exact
 natural frequency (a) cart displacement (b) load
 oscillation

- 5.31 Response of the gantry crane to 3rd order elliptic 90
 band-stop filter's filtered input with exact
 natural frequency (a) cart displacement (b) load
 oscillation
- 5.32 Power spectral density for the system with 91 unshaped input and exact natural frequency
- 5.33 Power spectral density for the system with exact 91
 natural frequency and filtered input using 1st
 order Butterworth low-pass filter (cut-off
 frequency at 25% of natural frequency)
- 5.34 Power spectral density for the system with exact 92 natural frequency and filtered input using 3rd order Butterworth low-pass filter (cut-off frequency at 25% of natural frequency)
- 5.35 Power spectral density for the system with exact 92 natural frequency and filtered input using 1st order Butterworth low-pass filter (cut-off frequency at 75% of natural frequency)
- 5.36 Power spectral density for the system with exact 93 natural frequency and filtered input using 3rd order Butterworth low-pass filter (cut-off frequency at 75% of natural frequency)

- 5.37 Power spectral density for the system with exact 93 natural frequency and filtered input using 1st order elliptic low-pass filter (cut-off frequency at 25% of natural frequency)
- 5.38 Power spectral density for the system with exact 94 natural frequency and filtered input using 3rd order elliptic low-pass filter (cut-off frequency at 25% of natural frequency)
- 5.39 Power spectral density for the system with exact 94 natural frequency and filtered input using 1st order elliptic low-pass filter (cut-off frequency at 75% of natural frequency)
- 5.40 Power spectral density for the system with exact 95
 natural frequency and filtered input using 3rd
 order elliptic low-pass filter (cut-off frequency at 75% of natural frequency)
- 5.41 Power spectral density for the system with exact 95 natural frequency and filtered input using 1st order Butterworth band-stop filter
- 5.42 Power spectral density for the system with exact 96 natural frequency and filtered input using 3rd order Butterworth band-stop filter
- 5.43 Power spectral density for the system with exact 96 natural frequency and filtered input using 1st order elliptic band-stop filter

- 5.45 Filtered input for 1st order Butterworth low-pass 98 filter designed with 20% error in exact natural frequency (cut-off frequency at 25% of the natural frequency)
- 5.46 Filtered input for 3rd order Butterworth low-pass 98 filter designed with 20% error in exact natural frequency (cut-off frequency at 25% of the natural frequency)
- 5.47 Filtered input for 1st order Butterworth low-pass 99 filter designed with 20% error in exact natural frequency (cut-off frequency at 75% of the natural frequency)
- 5.48 Filtered input for 3rd order Butterworth low-pass 99 filter designed with 20% error in exact natural frequency (cut-off frequency at 75% of the natural frequency)
- 5.49 Filtered input for 1st order elliptic low-pass filter 100 designed with 20% error in exact natural frequency (cut-off frequency at 25% of the natural frequency)

- 5.50 Filtered input for 3rd order elliptic low-pass filter 100 designed with 20% error in exact natural frequency (cut-off frequency at 25% of the natural frequency)
- 5.51 Filtered input for 1st order elliptic low-pass filter 101 designed with 20% error in exact natural frequency (cut-off frequency at 75% of the natural frequency)
- 5.52 Filtered input for 3rd order elliptic low-pass filter 101 designed with 20% error in exact natural frequency (cut-off frequency at 75% of the natural frequency)
- 5.53 Filtered input for 1st order Butterworth band-stop 102 filter designed with 20% error in exact natural frequency
- 5.54 Filtered input for 3rd order Butterworth 102 band-stop filter designed with 20% error in exact natural frequency
- 5.55 Filtered input for 1st order elliptic band-stop 103 filter designed with 20% error in exact natural frequency
- 5.56 Filtered input for 3rd order elliptic band-stop 103 filter designed with 20% error in exact natural frequency

- 5.57 Response of the gantry crane to 1st order 104 Butterworth low-pass filter's filtered input with cut-off frequency at 25% of natural frequency (filter consists 20% error in exact natural frequency)
- 5.58 Response of the gantry crane to 3rd order 105
 Butterworth low-pass filter's filtered input with cut-off frequency at 25% of natural frequency (filter consists 20% error in exact natural frequency)
- 5.59 Response of the gantry crane to 1st order 106 Butterworth low-pass filter's filtered input with cut-off frequency at 75% of natural frequency (filter consists 20% error in exact natural frequency)
- 5.60 Response of the gantry crane to 3rd order 107 Butterworth low-pass filter's filtered input with cut-off frequency at 75% of natural frequency (filter consists 20% error in exact natural frequency)
- 5.61 Response of the gantry crane to 1st order elliptic 108
 low-pass filter's filtered input with cut-off
 frequency at 25% of natural frequency (filter
 consists 20% error in exact natural frequency)

- 5.62 Response of the gantry crane to 3rd order elliptic 109
 low-pass filter's filtered input with cut-off
 frequency at 25% of natural frequency (filter
 consists 20% error in exact natural frequency)
- 5.63 Response of the gantry crane to 1st order elliptic 110
 low-pass filter's filtered input with cut-off
 frequency at 75% of natural frequency (filter
 consists 20% error in exact natural frequency)
- 5.64 Response of the gantry crane to 3rd order elliptic 111 low-pass filter's filtered input with cut-off frequency at 75% of natural frequency (filter consists 20% error in exact natural frequency)
- 5.65 Response of the gantry crane to 1st order 112
 Butterworth band-stop filter's filtered input (filter consists 20% error in exact natural frequency)
- 5.66 Response of the gantry crane to 3rd order 113
 Butterworth band-stop filter's filtered input (filter consists 20% error in exact natural frequency)
- 5.67 Response of the gantry crane to 1st order elliptic 114
 band-stop filter's filtered input (filter consists
 20% error in exact natural frequency)

- 5.68 Response of the gantry crane to 3rd order elliptic 115
 band-stop filter's filtered input (filter consists
 20% error in exact natural frequency)
- 5.69 Power spectral density for the system with 116 filtered input using 1st order Butterworth low-pass filter that consists 20% error in exact natural frequency (cut-off frequency at 25% of natural frequency)
- 5.70 Power spectral density for the system with 116 filtered input using 3rd order Butterworth low-pass filter that consists 20% error in exact natural frequency (cut-off frequency at 25% of natural frequency)
- 5.71 Power spectral density for the system with 117 filtered input using 1st order Butterworth low-pass filter that consists 20% error in exact natural frequency (cut-off frequency at 75% of natural frequency)
- 5.72 Power spectral density for the system with 117 filtered input using 3rd order Butterworth low-pass filter that consists 20% error in exact natural frequency (cut-off frequency at 75% of natural frequency)

- 5.73 Power spectral density for the system with 118 filtered input using 1st order elliptic low-pass filter that consists 20% error in exact natural frequency (cut-off frequency at 25% of natural frequency)
- 5.74 Power spectral density for the system with 118 filtered input using 3rd order elliptic low-pass filter that consists 20% error in exact natural frequency (cut-off frequency at 25% of natural frequency)
- 5.75 Power spectral density for the system with 119 filtered input using 1st order elliptic low-pass filter that consists 20% error in exact natural frequency (cut-off frequency at 75% of natural frequency)
- 5.76 Power spectral density for the system with 119 filtered input using 3rd order elliptic low-pass filter that consists 20% error in exact natural frequency (cut-off frequency at 75% of natural frequency)
- 5.77 Power spectral density for the system with 120 filtered input using 1st order Butterworth band-stop filter that consists 20% error in exact natural frequency

- 5.78 Power spectral density for the system with 120 filtered input using 3rd order Butterworth band-stop filter that consists 20% error in exact natural frequency
- 5.79 Power spectral density for the system with 120 filtered input using 1st order elliptic band-stop filter that consists 20% error in exact natural frequency
- 5.80 Power spectral density for the system with 121 filtered input using 3rd order elliptic band-stop filter that consists 20% error in exact natural frequency
- 5.81 Comparison in term of rise time, settling time 123 and overshoot of the system's response for filter designed with exact natural frequency
- 5.82 Comparison in term of rise time, settling time 126 and overshoot of the system's response for filter designed with 20% error in exact natural frequency
- 5.83 Level of vibration reduction (a) filter with exact 128 natural frequency (b) filter with 20% error in natural frequency

LIST OF SYMBOLS

A_p	-	maximum pass-band loss
A_{s}	-	minimum pass-band loss
A_1	-	magnitude response's minimum value when in pass
		band (dB)
BSF	-	band-stop filter
dB	-	decibels
g	-	gravitational acceleration
H_{∞}	-	H-infinity
kg	-	kilogram
LPF	-	low-pass filter
l	-	length of bar
т	-	mass
m_c	-	mass of the cart
m_L	-	mass of the load
Ν	-	Newton
n	-	filter order
PD	-	proportional-derivative

r	-	passband ripple
rad	-	radians
s, sec	-	second
Т	-	kinetic energy
T_{\max}	-	kinetic energy (maximum)
$T_n(\omega)$	-	Chebyshev polynomial of order <i>n</i>
U	-	potential energy
$U_{ m max}$	-	potential energy (maximum)
и	-	applied force at cart
ż	-	cart velocity
<i>х</i>	-	acceleration
<i>x</i> ₁	-	displacement of cart
heta	-	angle that exists between the bar and vertical axis
$\dot{ heta}$	-	bar's angle rate
ω_{c}	-	cut-off frequency
ω_n	-	natural frequency
ω_p	-	pass-band cut-off frequency
ω_{s}	-	stop-band cut-off frequency
ω_1	-	frequency at which the magnitude response first falls
		below minimum value of magnitude response when in
		pass band
&	-	and
%	-	percent

CHAPTER 1

INTRODUCTION

1.1 CRANE : OVERVIEW

In our environment, there is a necessity to transfer the things like equipment, things etc. from one place to another, whether there are far or not. In the workplace, for example, at construction or industrial sites, ports, railway yards and other similar locations, special equipment is needed to transport the materials. These materials are usually heavy, large and hazardous, which cannot be handling by workers. In order to make the work easier, cranes have been used to lift, move, position or place machinery, equipment and other large objects. There are many types of crane that been used for these purposes, such as tower crane, overhead crane, boom crane, gantry crane and others. Figures 1.1 and 1.2 shows examples of overhead crane and gantry crane, respectively.



Figure 1.1 Overhead crane



Figure 1.2 Gantry Crane

A crane consists of a hoisting mechanism (usually a hoisting line together with a hook) and a support mechanism. A cable with the load hanged on the hook is suspended from a point on the support mechanism. The support mechanism will moves the hanged load around the crane workspace, while the hoisting mechanism will lifts and lowers the load to prevent the obstacles in the path and locate the load at the desired location.

In handling the crane, safety is the most important point to consider while operating the crane. Hence, the crane must be operated in safe operating manner and procedures. For a crane operator, an experience causing by a crane's accidents can be frightening them. There are many cases and incident regarding on the crane's accidents. For example, in April 1993, the crane becomes unbalanced during two separate incidents at DOE sites in United States of America, which is in Hanford Site and Bryan Mound Site. The first incident occurred in 28th April 1993, where a crane becomes unbalanced while the boom was being lowered. The second incident occurred 2 days later, on 30th April 1993, which while loading the load, the weight of the load caused the crane to tip forward [1]. From these incidents, guidelines have been suggested in using the cranes. Some of the guidelines are:

- i. the weight of load must be checked.
- ii. crane operations should be supervised by qualified personnel.
- iii. crane operators must be familiar with their equipment.
- iv. crane operators must be trained and qualified to operate their equipment.

Although the guidelines have been sketched in order to prevent the accident, the other factors also must be considered so that the probability of accidents occurs is small or reduced at an acceptable value. There are many factors that have to be considered: the braking systems, hydraulic and pneumatic components, electrical equipment, operational aids, operating mechanisms, lifting devices, determining load weight, recognizing immediate and potential hazards, control systems and others. In term of control systems, the important issue is how to control the load swing. This is important in order to have a faster operation while maintaining the safety.

1.2 GANTRY CRANE

Generally, crane can be defined as a machine used for lifting and lowering a load vertically and moving it horizontally and that has a hoisting mechanism as an integral part of it. As mentioned before, a crane type has varies, depend on their application: automatic crane, cab-operated crane, cantilever gantry crane, floor-operated crane, gantry crane, jib crane, mobile crane, overhead traveling crane, power-operated crane, pulpit-operated crane, remote-operated crane, semigantry crane, wall-mounted crane and wall-mounted jib crane. In this project, the work will be focused on a gantry crane. Gantry crane is similar to an overhead crane, except that the bridge for carrying the trolley or trolleys is rigidly supported on two or more legs running on fixed rails or other runway. To implement the operation, the crane operator will seat inside the cart, and move the cart with the load hanged with it so that the load can achieve the desired location. A real crane may allow a cart movement of 80 to 90 meters [2], regarding on the desired load location.

1.3 MOTIVATION, RATIONAL, SIGNIFICANCE AND NEED FOR THE STUDY

From the previous works, it seems that most researchers have given a lot of efforts in developing a control algorithms and designing controllers that can be used and realized in nature. This includes the study related on how to reduce the vibration, especially in crane, where the controllers that been designed are mostly to control the load swing. Since this is relatively simple and well defined problem in dynamics and control, it is surprising that, it has not been solved exactly, where an exact solution is here understood to be a control strategy that guarantees complete success in a finite time. Most of the crane controllers that have been developed until now have been far from satisfactory. Once tested in actual operation, there found to be ineffective and thus were left unused. This may due to the standard control feedback strategies that are not well suited to this problem. Therefore, the problem of controller synthesis for a crane is still under consideration. Regarding on this matter, in this study, it seems interesting if multiple point of view can be taken in modeling the crane. For this purpose, gantry crane has been chosen in order to achieve the aim. This will involves in determining the relation between the cart' mass, load's mass and the load swing, in order to looking after the effect of the cart and load's mass to the load oscillation. Because the operation of the gantry crane is related with the movement of the cart and load, the effect that cause a vibration will be, whether from the acceleration that been applied at the cart, or the load and cart's inertia that been exists because the movement of these objects.

1.4 OBJECTIVES AND WORK METHODOLOGY

The objectives of this project can be divided as following:

- To obtain a mathematical model of the gantry crane for further analysis (including to study the system's natural response, transient behavior etc.).
- To verify the derived mathematical model through comparison with previous work and simulation on the model.
- To investigate the effects of system parameters such as load on the dynamics behavior of the system.
- To design and develop control algorithms for gantry crane based on filtering techniques.
- To investigate the performance of the control technique in term of vibration reduction and robustness.

In term of work methodology, it can be summarized as in Figure 1.3.



Figure 1.3 Work methodology

1.5 THESIS OUTLINE

This section will give an outlines of the structure of the thesis. The following is an explanation for each chapter.

<u>Chapter 2</u> discusses the previous work that been done around the world about the crane, in term of modeling, or designing the crane. Literature that been done will cover, for instance, modeling, control algorithm design and others. At the end of the chapter, conclusion regarding on previous work that have been surveyed will be showed.

<u>Chapter 3</u> deals with the gantry crane model, where the mechanical drawing and description related on it will be explained. In addition, the assumption and limitation that been added to the model will be described.

<u>Chapter 4</u> will discuss along the line regards to model the gantry crane, where mathematical expression will be derived and will be showed. For this work, consideration will be given to the gantry crane with one degree-of-freedom, which its cart and load movement is only along single axis. The derivation will lead to forming the state equation, and the critical aspects, for example, natural frequency will be focused. Furthermore, the derived equation will be compared with previous work and also simulation in order to validate the model. Other than that, the characteristic of gantry crane will be explored through simulation, where the simulation will take many factors that seems can give an effect to the gantry crane.

<u>Chapter 5</u> will discuss about control algorithm design for controlling gantry crane. In this topic, consideration is given to develop an algorithm to reduce oscillation of the load. The method that will be proposed is using command shaping via filtering technique, where the method is generally based on open-loop control. Analysis regarding on performance of designed controller will be conducted, and evaluation will be implemented.

<u>Chapter 6</u> contains conclusion regarding on the topics and recommendation for future works.

evaluate the performances of the gantry crane. With this implementation also, the other aspects perhaps can be observed in order to improve the design.

Finally, is to add other consideration in design, such as to take account the mass of the bar (which relates calculation regards of moment of inertia of the bar and its effect in the system); effect from environment (wind etc). For instance, with adding the mass of the bar, the moment inertia of the bar also need to be considered, which also have its role in load oscillation. As we know, since the body has a definite size and shape, an applied nonconcurrent force system may cause the body to both translate and rotate. This will lead to consideration of adding moment of inertia in calculation, which is a measure of the resistance body to angular acceleration. Regarding on this fact, it is quite beneficial if the other consideration is taken in our design.

REFERENCES

- Crane Safety. Issue No. 8: Occupational Safety Observer. http://tis.eh.doe.gov/docs/oso/oso93_08.html. 1993. August: 1 – 8.
- Hans Butler, Ger Honderd and Job Van Amerongen. Model Reference Adaptive Control of a Gantry Crane Scale Model. *IEEE Control Systems 1991*, 1991. 57 – 62.
- M.P. Cartmell. On the Need for Control of Nonlinear Oscillation in Machine Systems. *Meccanica International Journal of the Italian Association of Theoretical and Applied Mechanics AIMETA*, 2003. 38(2): 185 – 212.
- C.L. Teo, C.J. Ong and M. Xu. Pulse Input Sequences for Residual Vibration Reduction. *Journal of Sound and Vibration 1998*, 1997. 211(2): 157 - 177.
- Jia-Jang Wu, A.R. Whittaker and M.P. Cartmell. The Use of Finite Element Techniques for Calculating the Dynamic Response of Structures to Moving Loads. *Computers and Structures*, 1999. 78: 789 – 799.

- D.C.D. Oguamanam, J.S. Hansen and G.R. Heppler. Dynamic Response of an Overhead Crane System. *Journal of Sound and Vibration (1998)*, 1997. 213(5): 889 – 906.
- William J. O'Connor. Gantry Crane Control: A Novel Solution Explored and Extended. *Proceedings of the American Control Conference*. May 8-10, 2002. 250 – 255.
- Oliver Sawodny, Harald Aschemann and Stephan Lahres. An Automated Gantry Crane as a Large Workspace Robot. *Control Engineering Practice*, 2002. 10: 1323 – 1338.
- Y. Fang, W.E. Dixon, D.M. Dawson and E. Zergeroglu. Nonlinear Coupling Control Laws for a 3-DOF Overhead Crane System. *Proceedings of the 40th IEEE Conference on Decision and Control.* December 2001. 3766 – 3771.
- J.K. Pieper and B.W. Surgenor. Discrete Time Sliding Mode Control Applied to a Gantry Crane. *Proceedings of the 33rd Conference on Decision and Control*. December 1994. 829 – 834.
- Yannick Aoustin and Alexander Formal'sky. Simple Anti-Swing Feedback Control for a Gantry Crane. *Robotica*, 2003. 21: 655 – 666.

- William Singhoe, Lisa Porter, Michael Kenison and Eric Kriikku. Effect of Hoisting on the Input Shaping Control of Gantry Cranes. *Control Engineering Practice*, 2000. 8: 1159 – 1165.
- L.F. Mendonca, J.M. Sousa and J.M.G. Sa' da Costa. Optimization Problems in Multivariable Fuzzy Predictive Control. *International Journal of Approximate Reasoning*, 2004. 36: 199 – 221.
- J. Lowen Shearer, Arthur T. Murphy and Herbert H. Richardson. *Introduction to System Dynamics*. USA: Addison-Wesley Publishing Company. 1971
- Ernest O. Doebelin. System Dynamics: Modeling and Response. USA: Charles E. Merrill Publishing Company. 1972
- Francis S. Tse, Ivan E. Morse and Rolland T. Hinkle. *Mechanical Vibrations "Theory and Application"*. USA: Allyn and Bacon, Inc. 1978
- 17. William T. Thomson. *Theory of Vibration with Application*. Third Edition. USA: Prentice Hall. 1988
- Ferdinand P. Beer & E. Russell Johnston, Jr. Vector Mechanics for Engineers: Dynamics. Fifth Edition. USA: McGraw-Hill Book Company. 1988

- R.C. Hibbeler. *Engineering Mechanics: Dynamics*. Seventh Edition.
 USA: Prentice Hall. 1995
- S.U. Choi, J.H. Kim, J.W. Lee, Y. J. Lee, K.S. Lee. A Study on Gantry Crane Control using Neural Network Two Degree of PID Controller. *ISIE 2001, Pusan, Korea*, 2001. 1896 – 1900.
- E.M. Abdel Rahman, Ali H. Nayfeh, Ziyad N. Masoud. Dynamics and Control of Cranes: Overview. *Journal of Vibration and Control*, 2003. 9: 863 – 908.
- 22. Katsuhiko Ogata. *Modern Control Engineering*. Fourth Edition. USA:Prentice Hall Inc. 2002
- Nakano Michio, Mita Tsutomu. Seigyo Kiso Riron (Koten Kara Gendai Made). Japan: Shoukoudou Kabushiki Gaisha, 1998 (Japanese Language)
- 24. Z. Mohamed, M.O. Tokhi. Command shaping techniques for vibration control of a flexible robot manipulator. *Mechatronics*, 2004. 14: 69-90.
- 25. C. Britton Rorabaugh. *Digital Filter Designer's Handbook with C++ Algorithms*. Second Edition. USA: McGraw-Hill Company. 1997