# TEMPERATURE EFFECT OF IMPULSIVE MOTION IN AN AIR FLOATATION NOZZLE

GOH ZHEN HWEE

A project report submitted in partial fulfilment of the requirements for the award of the degree of Master of Mechanical (Engineering)

> Faculty of Mechanical Engineering Universiti Teknologi Malaysia

> > JUNE 2014

To my beloved wife

#### ACKNOWLEDGEMENT

In preparing this project dissertation, I was in contact with supervisors, researchers, academicians, practitioners and friends. They have contributed towards my understanding and thoughts.

I would like to express my most sincere appreciation to my project supervisor, Assoc. Prof. Dr. Kahar Osman, for guidance, advices, encouragement, motivation and friendship.

I also would like to express my most sincere gratitude to my wife for her supports throughout the whole thesis process.

My fellow postgraduate schoolmates, Soh Kian Jin, Fam Kok Yeh and Ng Chee Chung should also be appreciated for their helpful assistance and supports.

At last, I would like to express my gratitude to my employer, Ir. Sang Fat Chon, Ir. Lim Ah Bah and Ir. James Wong for the financial suppoer.

Thank you very much.

### ABSTRACT

This project examines the temperature effect of impulsive motion for an air floatation nozzle in three-dimensional using computation fluid dynamics approach. The nozzles were first modelled in two-dimensional and then extruded into three-dimensional. Some variations to the nozzle's geometry were made in order to study the effects of various geometry setups such as variation in web distance, lip separation and adding holes along the side bar. The results indicate that non-linear relationship between the web distance and temperature distribution across the web. The results also show that the pressure decreases non-linearly as the web distance increases. The results for lip separation case studies also show non-linear relationship between the lip separation and the temperature distribution across the web. Finally, additional holes along the side bar produces more uniform temperature and pressure distribution on the web.

### ABSTRAK

Projek ini mengkaji kesan suhu gerakan impulsif muncung pengapungan udara di tiga dimensi menggunakan pendekatan pengiraan cecair dinamik. Muncung mula dimodelkan dalam dua dimensi dan kemudian dibentukkan ke dalam tiga dimensi. Beberapa perubahan geometri muncung telah dibuat untuk mengkaji kesan penyusunan geometri dari beberapa variasi seperti dalam jarak web/jaringan, pemisah/pengasingan bibir dan menambah lubang di sepanjang sisi bar. Keputusan menunjukkan bahawa hubungan bukan linear antara jarak web dan taburan suhu di seluruh web. Keputusan juga menunjukkan bahawa tekanan menurun tidak linear kerana kenaikan jarak web. Keputusan bagi pemisahan bibir kajian kes juga menunjukkan hubungan linear antara pengasingan bibir dan taburan suhu di seluruh web. Akhir sekali, lubang tambahan di sepanjang bar sisi yang dihasilkan suhu lebih seragam dan taburan tekanan di web.

### **TABLE OF CONTENTS**

CHAPTER	TITLE			AGE
	DECLARATION			ii
	DED	ICATION		iii
	ACK	NOWLEDGEMENT		iv
	ABS'	TRACT		V
	ABS'	TRAK		vi
	TAB	LE OF CONTENTS		vii
	LIST	<b>FOF TABLES</b>		Х
	LIST	<b>FOF FIGURES</b>		xi
	LIST	Γ OF GRAPHS		xiii
	LIST	<b>FOF ABBREVIATIONS</b>		xiv
	LIST	Γ OF APPENDICES		XV
1	INTI	RODUCTION		
	1.1	Research Background		1
	1.2	Research Objective		3
	1.3	Problem Statement		4
	1.4	Scope of Research		5
	1.5	Significant of study		5
	1.6	Organization of Thesis		6
2	т тат			7
2		ERATURE REVIEW		7
	2.1	Introduction		7
	2.2	Impulsive Motion		7
	2.3	Air Impingement Dryer		11

RE	SEARCH METHODOLOGY	15
3.1	Introduction	15
3.2	Flow Chart of Research Methodology	16
	3.2.1 Project Flow Chart	16
	3.2.2 Simulation Flow Chart	17
3.3	Governing Equations	18
	3.3.1 Convective Heat Transfer	18
	3.3.2 Heat Equation	18
	3.3.3 Navier Stokes Equations	19
	3.3.4 Flow Assumptions	20
	3.3.5 Reynolds number	21
	3.3.6 Viscosity	22
	3.3.7 Computational Fluid Dynamics	23
3.4	Air Floatation Nozzle Parameters	29
3.5	Material Properties	30
3.6	Assumptions	31
3.7	Modeling of the nozzle	31
RE	SULTS AND DISCUSSIONS	33
4.1	Introduction	33
4.2	Analytical Results for Impulsive Motion	34
4.3	Impulsive Simulation	35
4.4	The Effects of Geometry Changes	37
	4.4.1 Web Distance – Temperature Profile	37
	4.4.2 Web Distance – Pressure Profile	40
	4.5.1 Lip Separation – Temperature Profile	44
	4.5.2 Lip Separation – Pressure Profile	46
	4.6 Nozzle with holes on side bar	49
	4.6.1 Velocity Profile	50
	4.6.2 Temperature Profile	51
	4.6.3 Pressure Profile	53

3

4

5	CON	NCLUSION AND RECOMMENDATIONS	
	5.1	Conclusion of Results	55
	5.2	Conclusion of Project	55
	5.3	Recommendation for Further Works	56
R	REFEREN	CES	57
APPENDICES			59

# LIST OF TABLES

TABLE NO.	TITLE	PAGE
1	Input data	13
3.1	Nozzle parameters	29
3.2	Air properties	30
3.3	Paper properties	30
4	Specification for both standard and modified nozzle	49
5	Results for 3D models	51

### LIST OF FIGURES

FIGURE NO.	TITLE	PAGE
1.1	Air impingement nozzles	3
1.2	Air impingement nozzle schematic diagram	4
2.1	Heat transfer profile on the web	11
2.2	Opposing impingement air bars	12
3.1	Residual plot	26
3.2	Convergence history	26
3.3	3D nozzle model from journal	29
3.4	2D nozzle model by solid work	31
3.5	3D nozzle model by solid work	32
3.6	Revised 3D nozzle to run in CFD	32
4.1	Temperature across the nozzle when $t = 0.15s$	35
4.2	Temperature across the nozzle when $t = 0.16s$	35
4.3	Temperature across the nozzle when $t = 0.17s$	36
4.4	Temperature across the nozzle became uniform when $t = 0.18s$	36
4.5	Temperature distribution across the web for distance of 10mm.	37
4.6	Temperature distribution across the web for distance of 20mm	37

4.7	Temperature distribution across the web for distance of 20mm.	38
4.8	Pressure Profile across the web for web distance =	40
	10mm	
4.9	Pressure Profile across the web for web distance = 20mm	40
4.10	Pressure Profile across the web for web distance = 25mm	41
4.11	Temperature distribution across the web for lip separation of 2mm.	43
4.12	Temperature distribution across the web for lip	43
7.12	separation of 3mm	-5
4.13	Temperature distribution across the web for lip	44
	separation of 4mm	
4.14	Pressure Profile across the web for lip separation of 2mm	46
4.15	Pressure Profile across the web for lip separation of	46
	3mm	
4.16	Pressure Profile across the web for lip separation of 4mm	47
4.17	Nozzle with holes along the side bar	49
4.18	Velocity Vector plot for standard nozzle	50
4.19	Velocity vector plot for nozzle with holes	50
4.20	Temperature contour plot on the web	51
4.21	Pressure contour plot on the web	53

xii

### LIST OF GRAPHS

GRAPH	NO.
-------	-----

## TITLE

#### PAGE

1	Temperature across channel	8
2.1	Specific solvent mass for different nozzle outlet velocities	14
4.2.1	Temperature across a channel with impulsive motion	34
4.4.1	Comparison of temperatures distribution across the web for distance for 10mm, 20mm and 25mm	38
4.4.2	Maximum temperature across the web with different web distances	39
4.4.3	Comparison of pressure distribution across the web for distance of 10mm, 20mm and 25mm	41
4.4.4	Maximum pressure across the web with different web distances	42
4.5.1	Comparison of temperatures distribution across the web for lip separation of 2mm, 3mm and 4mm	44
4.5.2	Maximum temperature across the web with different lip separation	45
4.5.3	Comparison of pressure distribution across the web for lip separation of 2mm, 3mm and 4mm	47
4.5.4	Maximum pressure across the web with different lip separation	48
4.6.1	Temperature distribution across the web for both standard nozzle and modified nozzle	52
4.6.2	Temperature distribution across the web for both standard nozzle and modified nozzle	53

# LIST OF ABBREVIATIONS

## Nomenclature

α	-	Thermal diffusivity
β	-	Thermal expansion coefficient
$C_p$	-	Specific heat capacity
k	-	Thermal conductivity
т	-	Mass flow rate
μ	-	Dynamic viscosity
η	-	Efficiency
ρ	-	Density
Q	-	Volumetric flow rate
Ø	-	Volume fraction
ν	-	Kinematics viscosity
x, y, z	-	Directions
U, v, w	-	Velocities in x, y an z
u	-	Fluctuating velocity
$\overline{u}$	-	Mean velocity $\tilde{u}$
ũ	-	Velocity vector (x, y,z)
Grad	-	Total derivative of $\phi$
Φ	-	Dissipation function
Μ	-	Dynamic viscosity
Re		Deeme 1 de march en
	-	Reynolds number

Unit		
Κ	-	Kelvin
Kg	-	Kilogramme
m³	-	Cubic meter
S	-	Second

## LIST OF APPENDICES

APPENDIX	TITLE	PAGE
A	Gantt Chart for MP1	59
В	Gantt Chart for MP2	59
С	User Defined Function for Impulsive Motion	60

### **CHAPTER 1**

### INTRODUCTION

#### 1.1 Research Background

Unsteady conditions of motion and heating of fluids is important in many applications which involve engineering fields of aerodynamics and hydrodynamics. For example, standard types of aerodynamics experiments have been carried out in shock tunnels for the past few decades, however, the experiment duration are generally too short that do not allow the solid surfaces to heat up to temperatures simulating the actual conditions. With the aid of transient development process, therefore design model configurations can be obtained. [1]

Analytical solutions of exact problems of physical relevance are very useful even though they are limited to simple geometries. Physical parameters governing the phenomenon can be identified clearly. Besides that, analytical solutions are the fundamental test cases for the verification of numerical methods. [2]

Analytical analysis of the problem above can be found in literature, Amilcare Pozzi and Renato Tognaccini [3] mentioned that incompressible flow arising in a two-dimensional channel when the imposed time law of the pressure gradient has a power expression. Due to the linearity of the Navier Stokes equations in the case of fully developed parallel flows, the solution in the case of arbitrary pressure gradient can be obtained by the aid of Taylor Series. Besides that, the temperature profile of the parallel wall involves the effects of the dissipation of kinetic energy. An exact analytical solution of the unsteady impulse thermo-fluid dynamic was also presented by them when thermal field in the fluid is coupled with the thermal field in the solid. The temperature and heat flux at the solid-fluid interface are analyzed as function of time and of the non-dimensional parameters governing the problem. [2]

Brereton and Jiang [4] investigated the convective heat transfer associated to unsteady laminar flows in pipe and channel with axial temperature gradient. The thermal energy equation can be determined analytically, yielding solutions for the instantaneous temperature field for arbitrary time unsteadiness in both the flow and the wall flux.

Numerical solution for unsteady heat transfer on boundary later growth was presented by Pop and Katagiri [5] in 1976. The study utilizes an alternative combination of an expansion method into power series of small time and a very efficient numerical method using the difference-differential method. Highlight of the study is the transient phenomena from initial flow to the final steady-state distribution.

An impulsive Falkner-Skan flow was presented by Harris, Ingham and Pop [1]. Analytical solutions for the simultaneous development of the thermal boundary layers are determined for both small and large times. These solutions are then compared using a very efficient finite-difference method.

To relate impulse flow to an industry application, drying process is considered in this paper. There are quite a number of drying methods, however in this paper, we are focusing on air impingement dryer as figure shown below:



**Figure 1.1 – Air Impingement nozzles** 

A complete model for the simulation of the drying process of a binary system in a modular air impingement dryer is presented by Aus, Durst and Rasziller [6]. The influence of the operating parameters on the drying process and the energy efficiency was investigated. However, in this paper, a single air impingement nozzle with impulsive motion will be investigated.

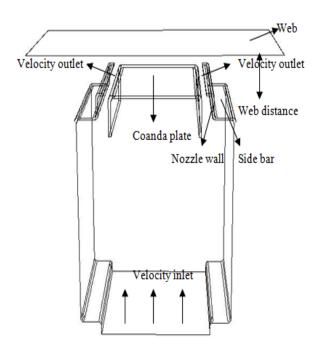
## **1.2** Research Objective

The objective of this thesis is:

- 1) To determine the temperature effect of an air floatation nozzle with impulsive motion
- 2) To investigate the effects of specific geometry changes to the nozzles
  - Change in web distance
  - Change in lip separation(Pressure outlet)

#### **1.3 Problem Statement**

The drying of moving substrates is the main procedure and the manufacture of many daily products such as paper and board, photographic films and pressure sensitive adhesives. It usually cost the most during the drying process and can have a major effect on the quality of the product [7]. A unique nozzle system for non-contact flotation drying, high porosity webs on its width up to 9 meters. Air is forced from the nozzle to form an air



cushion to support the web in order to convey it and the air will impinge onto the web in order to dry or the coatings instead of conveyor belts, air flotation provides advantages, such as: avoiding belt cleaning issues, reducing cost, accurate temperature and pressure profiles, and low maintenance [11].

Figure 1.2 – Air Impingement nozzle schematic diagram

Hot air will be ejected from the velocity inlet and then pass through the gap between the nozzle wall and the Coanda plate. The air is then impinged onto the web so as to carry out the drying or coating process.

The heat transfer problem is idealized as follows. The nozzle and the fluid are assumed to be initially at the same temperature. A thermal boundary layer is then produced by the sudden increase of the temperature of the fluid.

Temperature effect of the flow within the nozzle and web will be investigated. Besides that, several changes had been made in the geometry of the nozzle such as the distance between the nozzle and the web, lip separation of the nozzle and the magnitude of velocity inlet in order to investigate the effects on the overall flow. The aforementioned parameters and properties will be further discussed in later chapter.

### **1.4** Scope of Research

This thesis is to conduct a research on the application of impulsive flow in air flotation nozzle. By integrating the existing information of air flotation nozzle and impulsive flow, simulation of heat transfer of an air flotation nozzle with impulsive start-up and finally to verify and validate the simulation results. The scopes of this thesis are as follows:

- i) Analytical solution on a flow with impulsive motion past through a channel
- ii) Identification of impulsive start up properties.
- iii) Identification of air floatation nozzle design parameter.
- iv) Development of an single air floatation nozzle model.
- Numerical solution on the velocity, pressure and temperature of the nozzle

#### **1.5** Significant of study

- i) To obtain uniform temperature and pressure across the nozzle.
- ii) To improve the efficiency of the drying process

### **1.6** Organization of Thesis

To complete this project, the following steps are required to be implemented,

- i) Data collection from the published journals.
- ii) Design air flotation nozzle by using the aid of SolidWorks.
- iii) Simulation of 2D and 3D models by using the aid of Fluent
- iv) Setting of boundary conditions.
- v) Numerical analysis of temperature and pressure across the channel
- vi) Analysis of temperature and pressure profiles onto the web.
- vii) Analysis of uniform temperature and pressure across the nozzle.
- viii) Discussion
- ix) Conclusion

A weekly activity of this thesis has been presented in Gantt chart and appended in Appendix 1 and 2 for thesis 1 and 2 respectively.

#### REFERENCES

[1] Simon D. Harris, Derek B. Ingham and Ioan Pop [2002] "Unsteady heat transfer in impulsive Falkner-Skan flows: Constant wall temperature case".

[2] Amilcare Pozzi and Renato Tognaccini [2009] "Conjugated heat transfer in unsteady channel flows".

[3] Amilcare Pozzi and Renato Tognaccini [2008] "Thermo-fluid dynamics of the unsteady channel flow".

[4] G.J. Brereton and Y. Jiang [2006] "Convective heat transfer in unsteady laminar parallel flows".

[5] I. Pop and M. Katagiri [1976] "Unsteady Heat Transfer on Boundary Layer Growth at the Forward Stagnation Point".

[6] R. Aust, F. Durst and H. Raszillier [1997] "Modeling a multiple-zone air impingement dryer".

[7] Richard Wimberger [1995] "Curing Coated Webs with Flotation Dryers".

[8] Tom Puukila [2012] "Flotation Drying of Coated Grades"

[9] William R. Henry [2010] "Flotation Dryers – Expanding Applications for Air Foils".

[10] Noakes, C.J., Thompson, H.M., Gaskell, P.H., Lowe, D.C., Lowe, S. and Osborn, M.J. [2002] "Issues of pressure and web stability in the design of air flotation dryers", Paper Technology July 2002, Vol 43, No 6, Page 34-38

[11] Versteeg, H.K. & Malalasekera, W [1995] "An Introduction to Computational Fluid Dynamics: The Finite Volume Method. Harlow: Longman Scientific & Technical", Page 24.

[12] Koh, Y.Y. [2006] "EGM364 Combustion, heat and mass transfer", Inti International University College, Lecturer Notes, Lecture 7-13

[13] Symon, Keith [1971]. "Mechanics, Third Edition", Addison-Wesley. ISBN 0-201-07392-7.

[14] Streeter, V.L. [1966], "Fluid Mechanics", Example 3.5, McGraw-Hill Inc., New York

[15] Fluent User guide: "what is CFD?" [Online]. [Assessed 2008 March] Available from the web: <u>http://www.fluent.com/solutions/whatcfd.htm</u>

[16] Fluent 6.3 User Guide, Fluent Inc. September 2006. Chapter 1, Page 25 - 26.

[17] Op. cit [1] Chapter 1 page 4

[18] Prof. Gaskell, P. [2008] "MECH3825 Computational Fluid Mechanics", Leeds University, Mechanical Engineering Department, Lecturer Notes

[19] Op. cit [1] Chapter 1 page 2-3

[20] Fluent 6.3 User Guide, Fluent Inc. September 2006. Chapter 12 Turbulence modelling.

[21] Op. cit [1] Chapter 5 page 1