# MODELING AND CONTROL OF INDUSTRIAL SERVO PNEUMATIC ACTUATOR SYSTEM

MOHAMMAD HAMED HAJARI

UNIVERSITI TEKNOLOGI MALAYSIA

# MODELING AND CONTROL OF INDUSTRIAL SERVO PNEUMATIC ACTUATOR SYSTEM

### MOHAMMAD HAMED HAJARI

A project report submitted in partial fulfilment of the requirements for the award of the degree of Masters of Engineering (Electrical-Mechatronics & Automatic Control)

> Faculty of Electrical Engineering Universiti Teknologi Malaysia

> > JANUARY 2012

#### ACKNOWLEDGEMENT

In The Name of God.

I am grateful to God on His blessing in completing this project.

I would like to express my gratitude to my project supervisor, Professor Dr. Mohd. Fua'ad bin Hj. Rahmat for his encouragement, critics, and guidance. Under his supervision, many aspects regarding of this project has been explored, and with the knowledge, idea and support, this project can be completed.

Highest appreciation to my lecturers who had helped me in completing my studies. Without their continued passion in sharing knowledge, I would not be able to gain any understanding during study.

I also would like to express my gratefulness to my close friends Pouya, Hamed, Najib, Hazira, Nazila, Mahdi, mohammad and Amirmehdi for their caring and supports.

Finally I would like to dedicate this achievement to my parents, family and friends who have helped me directly or indirectly and sacrificed a lot in the completion of this study.

To my mother, father, brother and sister, Leila, Mohammad, Hamid and Hadis for their encouragement and blessing, support and caring...

#### ABSTRACT

Popularity of pneumatics is on the rise and they had received more attention since they offer many advantages over other type of force actuators such as costly effective, cleanliness and high power to weight ratio. Some of their engineering applications are included robotics, suspension system, haptic interface and etcetera. Servo pneumatic actuator system consists of a servo valve and a cylinder where its piston position needs to be controlled through servo valve as a commander in tracking desired trajectory along the stroke. The negative point of pneumatics which makes them difficult to control is their highly nonlinear behaviour. The objective of this study was to obtain mathematical model and control an industrial servo pneumatic system. Obtaining nonlinear mathematical model accurately to be used in controller design needs to determine all physical parameters of the real system which is very expensive and time consuming, to simplifying this procedure, model of system was analysed and obtained using system identification toolbox in Matlab. The system was excited with particular sine wave signal. Parametric approach using ARMAX structure was used to approximate the model. The best model was accepted based on the best fit criterion through SI toolbox. N-PID controller was designed for the model through the simulation. The results showed that N-PID controller provides better output than conventional PID controller. N-PID controller exhibits faster response to the system with desired transient error. But when the N-PID controller was applied on the real pneumatic system it showed very poor result because of existence of friction force. To improve the overall system output a friction compensator and a stabilizer attached to the N-PID controller. The system result illustrates that friction compensator and stabilized are very useful since they sufficiently enhance the controller performance. Self-tuning or robust controller beside of online system parameter estimation could be developed in future to increase the reliability of the controller.

#### ABSTRAK

Sejak kebelakangan ini, sistem pneumatik menjadi popular dan menjadi perhatian kerana sistem ini mempunyai kelebihan dari segi kosnya yang murah, bersih dan nisbah kuasa kepada berat yang tinggi. Sistem pneumatik banyak digunakan pada robotik, sistem suspensi, peratantaramukaan dan lain-lain. Sistem penggerak pneumatik servo terdiri daripada injap servo dan silender di mana kedudukan piston pada silender perlu dikawal melalui injap kawalan mengikut arahan isyarat masukan pada kedudukan trajektori yang di ingini. Kekurangan pada sistem pneumatik adalah pada ciri-ciri tidak linear yang sukar di kawal. Tujuan utama penyelidikan ini ialah untuk mendapatkan model matematik dan rekabentuk pengawal pada sistem penggerak pneumatik. Untuk mendapatkan model matematik tidak linear yang tepat, kesemua parameter fizikal pada sistem sebenar perlu diperolehi terlebih dahulu. Model dan parameter sistem di analisa dengan menggunakan pakej perisian Matlab dengan pengenalpastian sistem. Isyarat masukan yang digunakan untuk sistem ini ialah isyarat Sine. Pendekatan parametrik menggunakan struktur model ARMAX dunakan untuk menganggar model. Model terbaik adalah berdasarkan kepada kriteria titik terbaik. Pengawal N-PID direkabentuk untuk model melalui kaedah simulasi. Keputusan menunjukkan pengawal N-PID memberikan keputusan baik berbanding pengawal PID biasa. Pengawal N-PID menghasilkan sambutan fana yang pantas dengan ralat yang minimum pada simulasi. Tetapi apabila pengawal N-PID di gunapakai pada sistem sebenar, sambutan yang di hasilkan tidak memuaskan kerana daya geseran yang wujud pada piston silender. Untuk mengatasi masalah ini, pemampas geseran dan penstabil di gandingkan bersama dengan pengawal N-PID. Hasilnya di dapati pemampas geseran dan penstabil tersebut menambahbaik prestasi pengawal N-PID. Pelarasan sendiri atau pengawal robus boleh dibangunkan untuk meningkatkan prestasi pengawal pada kajian akan datang.

## **TABLE OF CONTENTS**

CHAPTER	TITLE		PAGE	
	DECLARATION			ii
	ACK	NOWL	EDGEMENTS	iv
	ABS	ABSTRACT		V
	ABS	ГRAK		vi
	TAB	LE OF (	CONTENTS	vii
	LIST	OF TA	BLES	xi
	LIST	OF FIG	GURES	xii
	LIST	OF AB	BREVIATIONS	xiv
1	INTF	RODUC	TION	
	1.1	Projec	t background	1
	1.2	Projec	t Objectives	3
	1.3	Projec	t Scope	4
	1.4	Projec	t Report Outline	4
2	LITE	CRATUI	RE REVIEW	
	2.1	System	n Model	5
	2.2	Contro	ol Strategies	7
	2.7	Summ	ary	8
3	PRO	JECT N	IETHODOLOGY	
	3.1	Mathe	matical Modeling	9
		3.1.1	Piston-Load Dynamic	10
		3.1.2	Dynamic of Pressure in the Cylinder	11
			Chambers	

	3.1.3	Valve Model	14
		3.1.3.1 Dynamic Analysis of Valve Spool	15
		3.1.3.2 Dynamic Analysis of Mass	18
		Flow of Air	
3.2	Trans	fer Function of Servo Pneumatic	19
		Actuator	
	3.2.1	Transfer Function Based on	19
		Physical Parameters	
	3.2.2	Transfer Function Using System	20
		Identification	
3.3	Contro	oller Design	27
	3.3.1	Nonlinear Function Construction Using	29
		offline Least Squares Estimation Method	
	3.3.2	N-IPD Controller for a Servo Pneumatic	30
		Actuator	
3.4	Summ	nary	31

## 4 **RESULTS AND DISCUSSION**

4.1	Model Simulation	33
	4.1.1 Nonlinear Mathematical Model Simulation	34
	4.1.2 Transfer Function of Linearized Equation	35
	Simulation	
4.2	Experimental Equipment	36
4.3	N-PID Controller	37
4.4	Closed Loop System Simulation using N-PID	41
	Controller	
4.5	N-PID Controller Implementation on Real Servo	41
	Pneumatic System	
4.6	Friction Compensator	42
4.7	Stabilizer	44
4.8	Complete Controller System Performance	45
4.9	Summary	48

# 5 CONCLUSION

5.1	Project Conclusion	49

5.2 Recommendations 50

REFERENCES	51

Appendices A-B	53
11	

## LIST OF TABLES

TABLE NO.	TITLE	PAGE
1.1	Electromagnetic motors and Hydraulics aspects	1
3.1	Some type of Valves and their application and functions	14
3.2	Ziegler-Nichols Method for PID Controller Tuning	28
4.1	Values of $K_P$ Corresponding to Variation of Step Size	38
B1	Rule Base Table for Both Positive and Negative Fuzzy	55
	Systems	

# LIST OF FIGURES

## FIGURE NO.

## TITLE

## PAGE

1.1	Piston-Cylinder Pneumatic Actuator	3
1.2	Rotary Type Pneumatic Actuator	3
3.1	Schematic Block Diagram of Typical Servo Pneumatic	9
	Actuator system	
3.2	Coulomb Friction + Stiction + Viscous Friction	10
3.3	Schematic Diagram of Valve Spool	15
3.4	Schematic of Effective Spool Displacement	17
3.5	ARMAX Model Structure	21
3.6	Multi Sine Shape Input Signal	22
3.7	Actual System Response	23
3.8	Introducing Data Sets to the System Id Toolbox	23
3.9	System Id Toolbox Window on MATLAB	24
3.10	Input and Output Data are Normalized and Divided by Two	024
3.11	Selecting ARMAX Model with Order of 3331	25
3.12	Estimated Output versus Actual Output	25
3.13	Residual Analysis; Autocorrelation and Cross correlation	26
3.14	Ideal and Parallel PID Controller Structure	27
3.15	Schematic Block Diagram of N-PID Controller	31
4.1	Simulation Model of Pneumatic System	34
4.2	Piston Movement when Multi Sine wave is used as input	34
	Signal to the Simulink Block	
4.3	Block Diagram of Open Loop System Based on Physical	35
	Parameter TF	
4.4	The Open Loop Response of Proposed TF for Pneumatic	35
	Actuator When Multi Sine Wave Used as Input	

4.5	Schematic Block Diagram of Servo Positioning Pneumatic	36
	System	
4.6	Experimental Setup to be Controlled in This Project	37
4.7	Proportional Directional Servo Valve, Pressure Sensor	37
	and Magnetic Transducer	
4.8	$K_P$ vs Error	39
4.9	N-PID Controller vs Two Conventional PIDs	40
4.10	Performance Comparison of N-PID with Conventional	40
	PIDs for Step with Size of 200mm	
4.11	Performance Comparison of N-PID with Conventional	40
	PIDs for Step with Size of 300 mm	
4.12	Closed Loop System Performance with N-PID Controller	41
	on MATLAB Simulink	
4.13	Actual Performance of N-PID Controller on the	42
	Pneumatic System	
4.14	Required Voltage to Compensate the Friction Effect	43
	Based on Piston Position	
4.15	Block Diagram of Friction Compensator	43
4.16	Oscillatory Response for Positive Set Point	44
4.17	Oscillatory Response for Negative Set Point	44
4.18	Block Diagram of Stabilizer	45
4.19	Complete Block Diagram of N-PID Controller with	46
	Friction Compensator and Stabilizer	
4.20	Performance of N-PID Controller with Friction	46
	Compensator and Stabilizer without Load	
4.21	Detail of Figure 4.20	47
4.22	Performance of N-PID Controller with Friction	47
	Compensator and Stabilizer with 10kg Load	
4.23	Detail of Figure 4.22	47
B1	Input Member Ship Function for Positive Fuzzy	54
B2	Output Member Ship Function for Positive Fuzzy	54
B3	Input Member Ship Function for Negative Fuzzy	55
B4	Output Member Ship Function for Negative Fuzzy	55

# LIST OF ABBREVIATIONS

ARX	- Auto-Regressive with Exogenous		
ARMAX	- Auto-Regressive Moving Average with Exogenous		
BJ	- Box-Jenkins		
ID	- Identification		
LS	- Least Squares		
N-PID	- Nonlinear Proportional Integral Derivative		
OE	- Output Error		
Р	- Proportional		
PI	- Proportional – Integral		
PD	- Proportional - Derivative		
PID	- Proportional Integral Derivative		
SI	- System Identification		

### **CHAPTER 1**

#### INTRODUCTION

#### 1.1 **Project background**

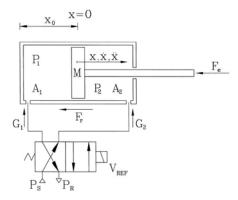
In industry particularly in the field of robotic and automation control many different type of force actuators namely electromagnetic motors, hydraulic systems and recently due to the development of control engineering and existence of low-cost and high-performance microprocessors, pneumatics are used. Traditionally, in industries electromagnetic motors and hydraulics have been utilized frequently. These types of force actuators have their own advantages and disadvantages. Table (1.1) shows some of their qualifications [1].

	Advantage	Disadvantage
Electromagnetic	1. Linear	1. Rotary motion
motors	characteristics	2. Transmission elements needed
	2. Fast	(high-speed and low-torque)
	3. Clean	3. High temperature in operation
Hydraulics	1. Linear motion (no	1. Noisy and makes work place
	interface needed)	hazards
	2. Can be directly	2. Dirty place because of oil
	connected to load	leakage
	3. Very powerful	

Table 1.1: Electromagnetic motors and Hydraulics aspects

The desire and favourable aspect of electromagnetic motors and hydraulic actuators is the ease of designing and implementing controller and it is the main reason that they have been employed by industries in servo application for a long time. In the past, though, pneumatics didn't take a part in servo application, but rather had been used to carry a payload between two fixed points, mostly two hard points of end of stroke. As mentioned, recent developments of control and computer science have been resulted in fast increasing demand of modern industries in employing pneumatics in servo systems [2]. Pneumatic actuators have many advantages over other types of mechanical energy converters. Pneumatics don't need any interface device to be coupled to the pay load, like hydraulics, while the electromagnetic motors because of their speed is high and their torque is not sufficient need an interface (gear box). In addition, pneumatics are clean, have easy and fairly cheap maintenance, and to be easily installed like electromagnetic motors while hydraulics are dirty by its nature that they work by oil so are not applicable in some certain clean environments. Moreover, the power to weight ratio of pneumatic actuators is significantly higher than equivalent electromechanical actuators [3]. According to what we just discussed, pneumatic systems can be used for some purposes such as working in dangerous environments, haptic interface, modern teleoperation force applications, and robotic industries [4, 5]. But unlike hydraulics and electromagnetic motors, pneumatic systems have highly nonlinear dynamic characteristics due to compressibility of air and highly nonlinear flow through pneumatic system components such as connecting tubes and valve orifice. Since gases are extremely compressible, there is a delay between the specifications of air flow travels in to the tube with that one which comes out from it at same time. On the other hand, in many application and design the distance between valve and piston is quite large in compare with the connecting tubes diameter and it will increase that time delay and the attenuation in mass flow rate produce some problems in pneumatics operations.

Pneumatic systems are divided into two main categories based on their performance characteristics such as fuel consumption, dynamic response, output power, and cost of design and manufacturing as: piston-cylinder and rotary actuators, which are shown by Figure (1.1) and (1.2) [6].



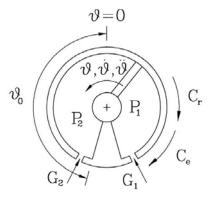


Figure 2.1: Piston-Cylinder Pneumatic Actuator

Figure 1.1: Rotary Type Pneumatic Actuator

Piston-Cylinder pneumatic actuators convert the power of medium (air) to linear reciprocating motion, but in vane rotaries the mechanical output energy is in the rotary motion form. Design and manufacturing of piston-cylinder is less expensive compare to rotary type and this fact caused the applying piston-cylinder actuators is more popular in industry than rotaries. On the other word, where simplicity and cost are prior, the piston cylinder is probably the best choice. But if fuel consumption is on the top of consideration, rotary type is indicated. Also have been shown the rotary servo has nearly twice the band pass of the piston cylinder servo [7, 8].

Based on all that mentioned before it can be learnt that pneumatics are a qualified alternative over hydraulics and electromechanical systems if there is a solution to tackle with its high nonlinearity characteristics. Thus, controller design and its implementation for pneumatic servo actuators is one of the challenging problems in control engineering.

#### **1.2 Project Objectives**

The project objectives are;

- i. To develop a mathematical model that describes the dynamics of servo pneumatic actuator system.
- ii. To design and implement a suitable controller for the real industrial servo pneumatic actuator.

#### **1.3 Project scope**

The scope of work in fulfilling the project objectives are;

- i. To study the characteristics and concepts of servo pneumatic actuator systems by referring to previous investigations through books and papers from past to date.
- ii. To construct a mathematical model of a complete servo pneumatic system to be applicable for computer simulation.
- iii. To obtain transfer function of hardware using system identification toolbox in MATLAB by processing input output data.
- iv. To design an appropriate nonlinear PID controller to improve the close loop system response.
- v. To design and implement friction compensator and stabilizer in order to extend the controller to tackle some uncertainties involved in real system to enhance system performance to satisfy all industrial requirements from a servo pneumatic actuator system.

### 1.4 **Project Report Outline**

This project report is organized in five chapters. The first chapter gives some overview of the system in this project and some useful information about its possible applications. Some literatures and previous researches with similar title have been reviewed by chapter 2. Chapter 3 covers the flow of the methodology and description of each procedure. Chapter 4 explains the results which obtained from experiment and discusses on the findings. And finally on chapter 5 summary of conclusion and recommendations are presented.

#### REFERENCES

- [1] M. Thomas, "ADVANCED SERVO CONTROL OF A PNEUMATIC ACTUATOR", *Ph. D. Thesis,* Ohio State University, 2003.
- [2] Ž. Šitum, "Nonlinear Mathematical Model of a Servo Pneumatic System", 9<sup>th</sup> International conference Antalya Turkey, 2005.
- [3] P. Beater (2007). Pneumatic Drives (System Design, Modeling and Control), Verlag Berlin Heidelberg: Springer.
- [4] Edmond Richer and Yildirim Hurmuzlu, 2000. A High Performance Pneumatic Force Actuator System (Part 1 - Nonlinear Mathematical Model). ASME Journal of Dynamic Systems Measurement and Control, Vol. 122, No.3, pp. 416-425.
- [5] Edmond Richer and Yildirim Hurmuzlu,2000. A High Performance Pneumatic Force Actuator System Part 2 - Nonlinear Controller Design .ASME Journal of Dynamic Systems Measurement and Control, Vol. 122, No.3, pp. 426-434.
- [6] Sorli, L. Gastaldi, E. Codina, S. de las Heras, 1999. Dynamic analysis of pneumatic actuators. Elsevier Science.
- [7] Hazem I. Ali, Samsul Bahari B Mohd Noor, S. M. Bashi, M. H. Marhaban, 2009. A Review of Pneumatic Actuators (Modeling and Control). Australian Journal of Basic and Applied Sciences, 3(2): 440-454.
- [8] Hazem I. Ali, Samsul Bahari B Mohd Noor, S.M. Bashi, Mohammad Hamiruce Marhaban, 2009. Mathematical and Intelligent Modeling of Electropneumatic Servo Actuator Systems. Australian Journal of Basic and Applied Sciences, 3(4): 3662-3670. M.
- [9] G.Kothapalli, and M. Y. Hassan (2008). Design of a Neural Network Based Intelligent PI Controller for a Pneumatic System. IAENG International Journal of Computer Science, 35(2).
- [10] M. F. Rahmat, S. N. S. Salim, A. A. M. Faudzi, Z. H. Ismail, S.I.Samsudin, N.H.Sunar, and K.Jusoff (2011). Non-linear Modeling and Cascade Control of an Industrial Pneumatic Actuator System. Australian Journal of Basic and Applied Sciences, 5(8): 465-477.

- [12] Ž. Šitum, J. Petrić, "Modeling and Control of Servopneumatic Drive", *Strojarstvo*, Vol. 43, No. 1-3, pp. 29-39, 2001.
- [13] F. Najafi and M. Fathi, 2007. Position Control of an Experimental Servo Pneumatic Actuator, Using Sliding Mode Control. K.N. Toosi University of Tech publication.
- [14] Z. Situm, D. Pavkovic, B. Novakovic, 2004. Servo Pneumatic Position Control Using Fuzzy PID Gain Scheduling. Transactions of the ASME Vol. 126, JUNE 2004.
- [15] J. Yu, J. Liu, X. Liu, Intelligent Fuzzy PID Control of Pneumatic Position Servo System, Vol. 186 (2011) pp 100-104Online available since.
- [16] N. R. Bin Abdullah "Modeling and Control of Magnetic Levitation System", *Master Project Report*, University Teknologi Malaysia, 2011.
- [17] Ziegler, J.G and Nichols, N. B. (1942). *Optimum settings for automatic controllers*. Transactions of the ASME. **64**. pp. 759–768.
- [18] K. Johan Astrom, B. Wittenmark (1989), Adaptive Control, America: Addison-Wesley.