

NUMERICAL SOLUTION OF CONVECTIVE HEAT TRANSFER IN A POLAR  
CAVITY

ALFRED SAGAYAM A/L VILAVANDRAN

A thesis submitted in fulfilment of the  
requirements for the award of the degree of  
Master of Engineering (Mechanical Engineering)

Faculty of Mechanical Engineering  
Universiti Teknologi Malaysia

JAN 2012

## ACKNOWLEDGEMENT

The successful completion of my project can be attributed to the combined efforts made by and the contribution made in one form or the other, by the individuals I hereby acknowledge. First of all, I express our heartfelt gratitude and thanks to my supervisor Prof Amer Nordin Darus, for encouraging me to move ahead and for being with me right from the beginning of the project and guiding me at every step.

I am grateful to my fellow course meets for his timely help. And last but not the least I also thank my parents and friends for being with me all the time and giving me moral support.

## ABSTRACT

Numerical solution of 2-D polar cavity is very less investigated analytically, numerically and experimentally. The investigation is mainly to understand the flow phenomena in the domain. One of the difficulties that occur for the numerical investigation is domain setup for complex geometry. The numerical solution for polar cavity is made possible by ANSYS FLUENT. Steady state incompressible ideal gas is considered for simplicity and other mechanical and thermal properties of fluid are constant with respect to temperature and pressure. The cavity's stationary walls such as inner radial wall and side wall kept as isothermal wall, while outer radial wall is set in motion in circumferential direction. The physical characteristic of flow phenomena in the polar cavity is analysed for different Reynolds numbers and different angles ( $30^{\circ}$ ,  $60^{\circ}$  and  $90^{\circ}$ ). Based on the mean velocity results, convective heat transfer coefficient in the driven curve cavity is computed. At end of the study, it is expected the convective coefficient increases with respect to the Reynolds numbers and cavity angle. The results are verified based on the numerical solution found in the in the published literature.

## ABSTRAK

Penyelesaian secara berangka, analitik mahupun secara ujikaji masalah rerongga kutub memang agak kurang diberi perhatian. Penelitian ini dilakukan adalah untuk memahami gelagat aliran di dalam rerongga tersebut. Salah satu masalah yang dihadapi dalam penyelesaian secara berangka ialah dalam membangunkan domin penyelesaian geometri yang kompleks ini. Penyelesaian berangka masalah aliran di dalam rerongga kutub ini telah mampu dilaksanakan dengan ANSYS FLUENT. Untuk memudahkan penyelesaiannya aliran bendalir dianggap sebagai gas unggul lagi tidak boleh mampat dengan sifat mekanikal dan termalnya kekal malar terhadap suhu dan tekanan. Semua dinding rerongga dianggap isothermal kecuali dinding atas yang bergerak. Ciri-ciri fizikal fenomena aliran di dalam rerongga kutub ini dianalisis pada pelbagai magnitud nombor Reynolds dan sudut bukaan yang berlainan. Berdasarkan hasil halaju purata dinding yang bergerak, koefisien pemindahan haba olakan di dalam rerongga kutub tersebut ditentukan secara berangka. Kajian ini menunjukkan bahawa nilai koefisien pemindahan haba olakan di dalam rerongga kutub bertambah dengan pertambahan magnitud nombor Reynolds dan sudut bukaan rerongga tersebut. Keputusan ini disahkan kebenarannya melalui perbandingan nilai koefisien pemindahan haba olakan masalah serupa yang diberi dalam literatur yang telah diterbitkan.

## TABLE OF CONTENTS

CHAPTER	TITLE	PAGE
	<b>TITLE PAGE</b>	<b>i</b>
	<b>DECLARATION</b>	<b>ii</b>
	<b>ACKNOWLEDGEMENT</b>	<b>iii</b>
	<b>ABSTRACT</b>	<b>iv</b>
	<b>ABSTRAK</b>	<b>v</b>
	<b>TABLE OF CONTENTS</b>	<b>vi</b>
	<b>LIST OF TABLES</b>	<b>ix</b>
	<b>LIST OF FIGURES</b>	<b>x</b>
	<b>LIST OF SYMBOLS</b>	<b>xii</b>
<b>1</b>	<b>INTRODUCTION</b>	<b>1</b>
	1.1 Overview	1
	1.2 Background Study	2
	1.3 Problem Statement	3
	1.4 Objective	3
	1.5 Significance of the Project	3
	1.6 Research Scope	4
	1.7 Research Methodology	5
<b>2</b>	<b>LITERATURE REVIEW</b>	<b>6</b>
	2.1 Introduction	6
	2.2 Driven Cavity Problem	6
	2.2.1 Polar Cavity Flow	9
	2.3 Thermal Effects in Driven Cavity	11
	2.4 Meshing Selection	12

	2.5	Conclusion	13
<b>3</b>		<b>METHOLODOLOGY</b>	<b>14</b>
	3.1	Introduction	14
	3.2	Defining Governing Differential Equations	14
	3.3	Integral Forms of the General Transport Equations	17
	3.4	Data Input	18
	3.5	Boundary Conditions	19
	3.6	Problem Solving With CFD	20
	3.6.1	Geometry Modeling	20
	3.6.2	Problem Set Up in FLUENT	23
	3.7	Grid Independent	26
	3.8	Data Validation	27
	3.9	Data Output	27
	3.10	Computation of Convective Heat Transfer	28
	3.11	Conclusion	29
<b>4</b>		<b>RESULT and DISCUSSION</b>	<b>30</b>
	4.1	Introduction	30
	4.2	Pre-Processing	30
	4.3	Results	32
	4.3.1	Input Data	32
	4.3.2	Output Data	34
	4.3.2.1	Velocity Profile Along the Centerline	34
	4.3.2.2	Transverse Velocity Vector of Polar Cavity	37
	4.3.2.3	Streamline Patterns, Vorticity Contours and Temperature Contours for Polar Cavity	39
	4.4	Coefficient of Convective Heat Transfer	44
	4.5	Conclusion	44
<b>5</b>		<b>CONCLUSION</b>	<b>46</b>
	5.1	Conclusion	46

5.2	Recommendation	46
-----	----------------	----

<b>REFERENCES</b>	<b>48</b>
-------------------	-----------

**LIST OF TABLES**

<b>TABLE NO.</b>	<b>TITLE</b>	<b>PAGE</b>
3.1	Reference velocity for various Reynolds numbers	19
3.2	Steps in geometry modeling	21
3.3	Steps in setting up the problem for FLUENT solution	24
4.1	Pre-processing information data	31
4.2	Speed of moving wall	33
4.3	Thermal boundary condition	33
4.4	Coefficient of convective heat transfer in the Cavity	44



## LIST OF FIGURES

FIGURE NO.	TITLE	PAGE
2.1	Lid driven cavity flow	7
2.2	Polar driven cavity flow	10
2.3	Interior angles of quad mesh and triangular mesh	12
3.1	Polar cavity	15
3.2	Geometry modeling with two radial and two circumferential walls	22
3.3	Mesh edges selection to set mesh intervals	22
3.4	Complete geometry modeling with mesh generation and boundary condition	23
3.5	Grid verification and quality checking	25
3.6	Viscous model and fluid material selection	25
3.7	Solver setting up	26
3.8	Result display	27
4.1	Velocity profiles along centerline $\theta_{\max}=\pi/6$	35
4.2	Velocity profiles along centerline $\theta_{\max}=\pi/3$	35
4.3	Velocity profiles along centerline $\theta_{\max}=\pi/2$	36
4.4	Transverse velocity vector for polar $\theta_{\max}=\pi/6$	37
4.5	Transverse velocity vector for polar $\theta_{\max}=\pi/3$	38
4.6	Transverse velocity vector for polar $\theta_{\max}=\pi/2$	38
4.7	Comparison of streamline contours, vorticity contours thermal contours for polar $\theta_{\max}=\pi/6$	41
4.8	Comparison of streamline contours, vorticity contours thermal contours for polar $\theta_{\max}=\pi/3$	42

4.9	Comparison of streamline contours, vorticity contours thermal contours for polar $\theta_{\max}=\pi/2$	43
-----	---	----

## LIST OF SYMBOLS

$\alpha$	-	Thermal diffusivity
$\beta$	-	Coefficient of thermal expansion
$c_p$	-	Specific heat at constant pressure
$\theta$	-	Cavity angle
$\nu$	-	Viscosity
$\rho$	-	Density
$\Phi$	-	Dissipation
$D$	-	Hydrodynamic length
$h$	-	Coefficient of convection
$k$	-	Coefficient of conduction
$L$	-	Reference length; $L=R_1$
$T$	-	Dimensionless temperature
$T_w$	-	Wall temperature
$T_1$	-	Entrance temperature
$R_1$	-	Inner radius
$R_2$	-	Outer radius
$V_w$	-	Speed of the outer radial wall
$u$	-	Radial component of velocity
$v$	-	Tangential component of velocity
$l$	-	Characteristic length
$z$	-	Z-axis
$y$	-	Y-axis
$x$	-	X-axis
$Re$	-	Reynolds number
$AP$	-	Aspect ratio
$Pr$	-	Prandtl number

<i>Ec</i>	-	Eckert number
<i>ADI</i>	-	Alternating-direction implicit
<i>N-S</i>	-	Navier-stokes
<i>FDM</i>	-	Finite Different Methods and
<i>FEM</i>	-	Finite element methods

## **CHAPTER 1**

### **INTRODUCTION**

#### **1.1 Overview**

Computational Fluid Dynamics, (CFD) become upsurge of interest by means of high performance computing hardware and user friendly interfaces have led to diverge of fluid analyzing. The CFD techniques have implemented into design, research and development, and manufacture of aircraft and automotive. As the CFD is appropriate a vital component in the flow analysis, driven cavity flow have always been the attention due to the simplicity of geometry and boundary condition. Contempt its simplicity, it is helped to shows attractive flow features as the geometries and Reynolds numbers varies. With available tremendous of computer-based simulation encouraging to analyze the different geometries and varies Reynolds numbers to present different flow features with high accuracy and numerical efficiency. The majority of cavity driven flows for steady incompressible flow were done by numerical studies. These shows discrepancies in studies, however, this study can be continued to computational of heat transfer in the cavity.

## 1.2 Background Study

The 2-D incompressible cavity flow has been studied numerically and experimentally. The 2-D cavity might be square, rectangular, triangular, and cone shapes. It can be polar cavity. They are numerous important engineering application this gadget. Most of the literatures presented the numerical studies about physical flow patterns inside the cavity for various Reynolds numbers. Among the literatures are Erturk E. and Gokcol O. (2005), Henri C.H and Saad M. (2005), Baytas A.C. and Liaqat A. (2000), Glowinski R., Guidoboni G. and Pan T.-W (2005), Xu H., Zhang C. and Barron R. (2005), Cheng M. and Hung K.C. (2005), Povitsky A. (2005), where each of them present different geometries and the results with different numerical approach.

In 1977, Ghia U. Ghia K.N and Studeruss C.J, computed 3-D laminar incompressible flow in straight polar cavity by numerical method. In the study some assumption have been done which is polar duct's axial flow direction the governing differential equations are parabolic, while for cross-flow the equation remain as elliptic for the computation of 3-D polar duct. To accomplish the assumption, they have made some limitation on their analysis such that entrance flow Reynolds number cannot be too low and the axial velocity must not be negative values everywhere.

The applications of the cavity flow are turbo machinery flow in the blade passages, heat exchangers and blood flow in the human arterial system. In engineering, examiner really wants to study the fluid motion in a cavity in order to optimize the effectiveness of flow in that cavity. It shows that the driven cavity flow analysis is very important in our daily life.

### **1.3 Problem Statement**

Polar cavity driven flow is very less investigated due to its complex geometry and complex mathematical modeling. Eventually, investigator cannot examine the fluid motion inside of driven cavity. Based on literature, the last was analyzed by Ghiak.U, et al. (1977) and it is not examined for high Reynolds numbers.

Incompressible flow in polar cavity became very subjective to study the fluid motion in engineering application such as in pump's impellers, turbo machinery's blade passages, rotating heat exchangers and blood flow in the human arterial system and etc. Besides, the coefficient of forced convective heat transfers cannot to be compute due to unknown mean velocity at cavity.

With these considerations, a study in polar driven cavity is conducted to study the flow physical and to calculate mean velocity of fluid inside cavity and the rate of heat transfer between fluid and the wall.

### **1.4 Objective**

The objective of this study is to analyze numerically the effect fluid flow physical; especially the mean velocity of fluid on coefficients of convective heat transfer for varies Reynolds no and cavity angles.

### **1.5 Significance of the Project**

One of the important of this project is to study the physical properties of flow and understand the rate of heat transfer from body to fluid. Most of today's technologies operates with higher frequency (rotation or translation or even in microprocessor) which dissipate heats. The faster heat release system helps a system

to operate effectively and efficiently. Besides, this numerical study contributes to development department of design to create very effective channel to release heat from particular engineering application, especially turbo machinery and motors. With this particular design concept will improve the reliable of component or equipment or an integrated system. To succeed the goal at first have to study the flow physical and the rate of heat transfer at polar cavity.

## 1.6 Research Scope

This project focuses on the numerical study of convective coefficient of heat transfer in polar cavity, where the flow profile is taken for consideration to achieve the main objective. The research scopes of this project are as follow;

Design criteria

- I. Development of polar cavity as discussed:  
Cavity angles ( $30^{\circ}$ ,  $60^{\circ}$ ,  $90^{\circ}$ ); dimensionless radius is constant.
- II. Identify data input for the software (Gambit and Fluent)  
Flow analysis:
- III. Analyze the flow physical characteristic; the velocity profile in x and y direction as Reynolds numbers increases.
- IV. Analyze the flow in regions with corners and curved boundaries.  
Thermal analysis:
- V. Analyze temperature distribution inside the cavity
- VI. Numerically compute the coefficient of heat transfer for various Reynolds number and cavity angles.



## 1.7 Research Methodology

To complete this project, few important steps need to be followed. The steps are:

- I. Data collection
- II. Polar cavity design for angle ( $30^{\circ}$ ,  $60^{\circ}$ ,  $90^{\circ}$ )
- III. Reynolds number ranging (0, 100, 200, 500, 800, 1000, 1200, 1500)
- IV. Boundary condition settings
- V. Development of meshing
- VI. Identify the convergence of results
- VII. Analyse velocity profile, stream line and eddy at corners
- VIII. Do numerical analysis to calculate mean velocity in cavity
- IX. Analyse thermal distribution in cavity
- X. Compute the coefficient convective heat transfer
- XI. Repeat steps, III to X for greater angle
- XII. Discuss the flow phenomena inside cavity, thermal distribution and at last not least the effect of Reynolds number and cavity angle on coefficient of convective heat transfer
- XIII. Conclusion

## REFERENCES

- A.Liaqat and A.C.Baytas (2000). Characteristic of conjugate free convection in a corium pool formed during a serve reactor accident. *Science Direct*.
- A.Liaqat and A.C.Baytas (2001). Numerical comparison of conjugate and non-conjugate natural convection for internally heated semi-circular pools. *Science Direct*.
- A.Povitsky (2005). Three-dimensional flow in cavity at yaw. *Science Direct*.
- C.A. Long, N.D.D. Miché, P.R.N. Childs (2007). Flow measurements inside a heated multiple rotating cavity with axial throughflow. *Science Direct*.
- C.H.Bruneau and M.Saad (2005). The 2D lid-driven cavity problem revisited. *Science Direct*.
- C. Lei, L. Cheng, K. Kavanagh (1999). A finite difference solution of the shear flow over a circular cylinder. *Science Direct*.
- E.Erturk (2005). Discussions on driven cavity flow and steady solutions at high Reynolds numbers. *Science Direct*.
- E.Erturk and O.Gokcol (2005). Fine grid benchmark solutions of triangular cavity flow. *International Journal of Numerical Methods for Heat & Fluid Flow*.
- Frank P. Incropera. *Heat Transfer*. 6th Edition International Student Version. Wiley. 2011.
- Hatice Mercan, Kunt Atalik (2008).Vortex formation in lid-driven arc-shape cavityflows at high Reynolds numbers. *Science Direct*.
- H. Xu, C. Zhang and R. Barron (2005). Numerical simulation of cavity flows based on transformed equations. *Science Direct*.
- Lin Mu, Xiu Ye (2011). A finitevolumemethod for solving Navier–Stokes problems. *Science Direct*.

- M. Cheng and K.C. Hung (2005). Vortex structure of steady flow in a rectangular cavity. *Science Direct*.
- R.Glowinski, G.Guidoboni and T.W.Pan (2005). Wall-driven incompressible viscous flow in a two-dimensional semi-circular cavity. *Science Direct*.
- Suhas N. Patankar. *Numerical Heat Transfer and Fluid Flow*. Taylor& Francis. 1980.
- Timothy J. Baker (2005). Mesh generation: Art or science? *Science Direct*.
- U.Ghia and K. Ghia (1977). Three-dimensional laminar incompressible flow in straight polar ducts. *Science Direct*.
- U.Ghia and R.K.Goyal (1976). A Study of Laminar Incompressible recirculating flow in a driven cavity of polar cross-section. *In Numerical~Laboratory Computer Methods in Fluid Mechanics*.
- Yunus A. Cengel & Michael A. Boles. *Thermodynamic*. Fourth Edition. McGraw-Hill.